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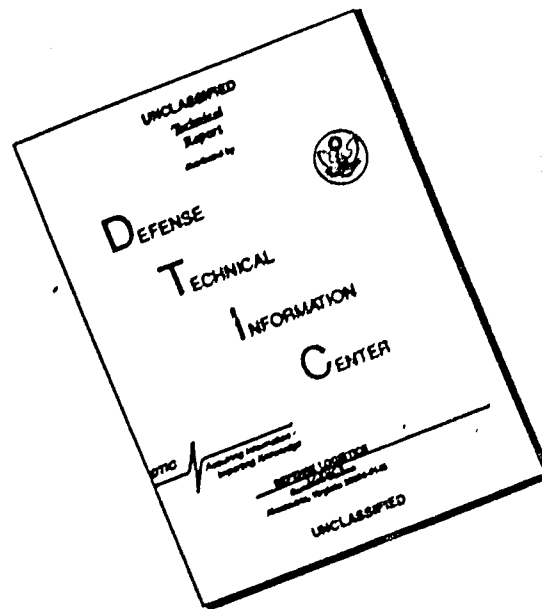


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# NEIL AIRCRAFT CORPORATION

## LIBERT-ST. LOUIS MUNICIPAL AIRPORT

DETAILED FINAL REPORT OF RESEARCH ON  
HIGH-SPEED ROTARY-PIED WING AIRCRAFT

VOLUME VII

SAMPLE AIRCRAFT POWER PLANT AND TEST ANALYSIS

OFFICE OF NAVAL RESEARCH, MEMPHIS BRANCH  
PROJECT NR 250-001 CONTRACT NR 250-001

Report 1905-1

Serial 15

29 December 1950

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Enclosure (5) to  
MAC Letter 2136-70P-1756

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DETAILED FINAL REPORT OF RESEARCH ON

HIGH SPEED ROTARY-FINED WING AIRCRAFT

VOLUME VII

SAMPLE AIRCRAFT POWER PLANT AND DUCT ANALYSIS

SUBMITTED UNDER Contract N9onr-64901 to the Office of Naval Research,  
Amphibious Branch, Project NR 251-001

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MODEL 78

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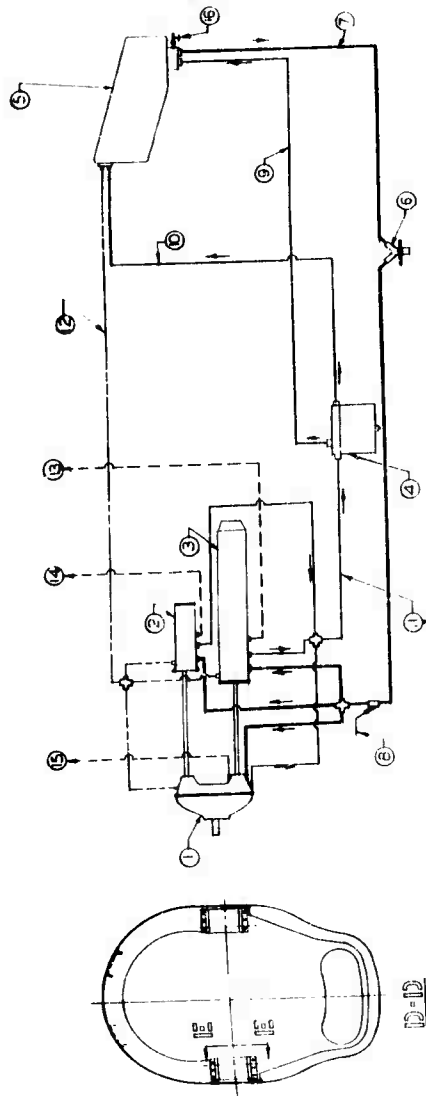
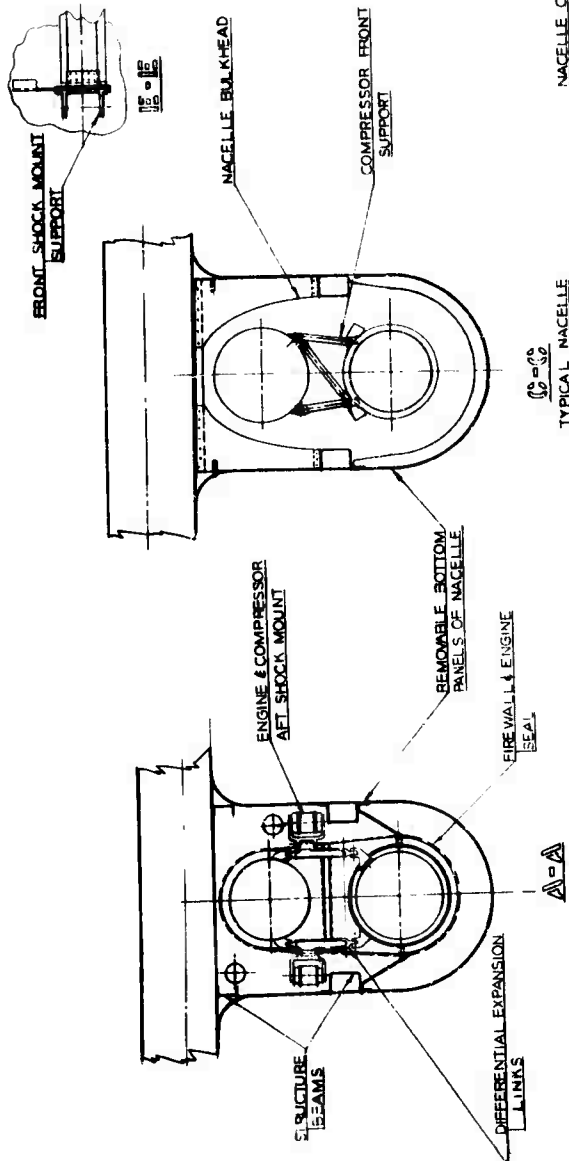
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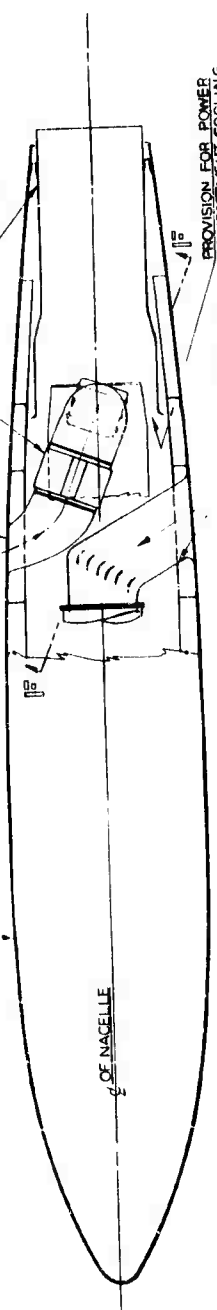
- 1 REDUCTION GEAR ASSEMBLY
  - 2 WESTINGHOUSE 18XB2B COMPRESSOR
  - 3 ALLISON-T38 TYPE ENGINE
  - 4 OIL COOLER
  - 5 OIL TANK
  - 6 DRAIN VALVE
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  - 8 OIL IN TEMP
  - 9 OIL COOLER BYPASS
  - 10 RETURN OIL COOLER TO TANK
  - 11 HOT OIL RETURN TO COOLER
  - 12 OIL TANK VENT
  - 13 ENGINE OIL PRESS.
  - 14 COMPRESSOR OIL PRESS.
  - 15 REDUCTION GEAR OIL PRESS.
  - 16 SUMP DRAIN VALVE
- TOTAL CAP - 17.5 GAL.  
OIL CAP - 4.5 GAL.

SHOWING FRONT SHOCK MOUNT SUPPORT & BULKHEAD

NACELLE CONTOUR

TYPICAL NACELLE BULKHEAD SHOWN

OIL SYSTEM - SCHEMATIC



PLAN VIEW

WING NOT SHOWN

WESTINGHOUSE 18XB2B COMPRESSOR

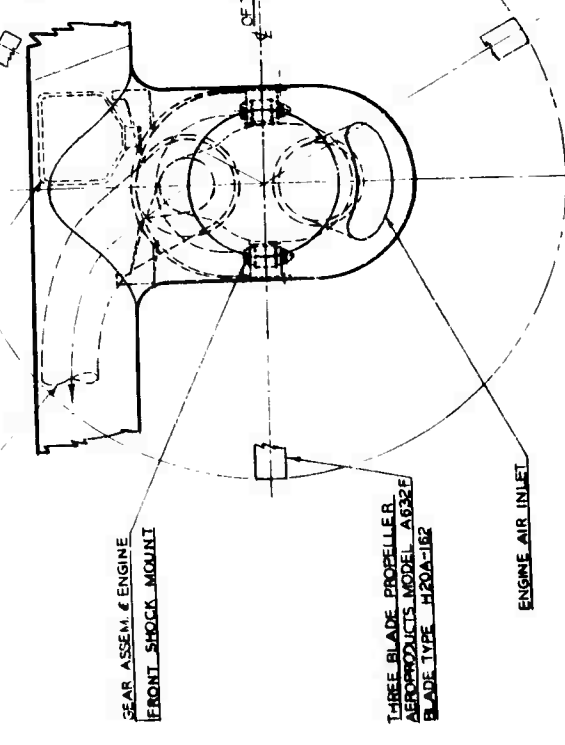
ALLISON-T38 TYPE ENGINE

REDUCTION GEAR ASSEMBLY DWG. S-1517

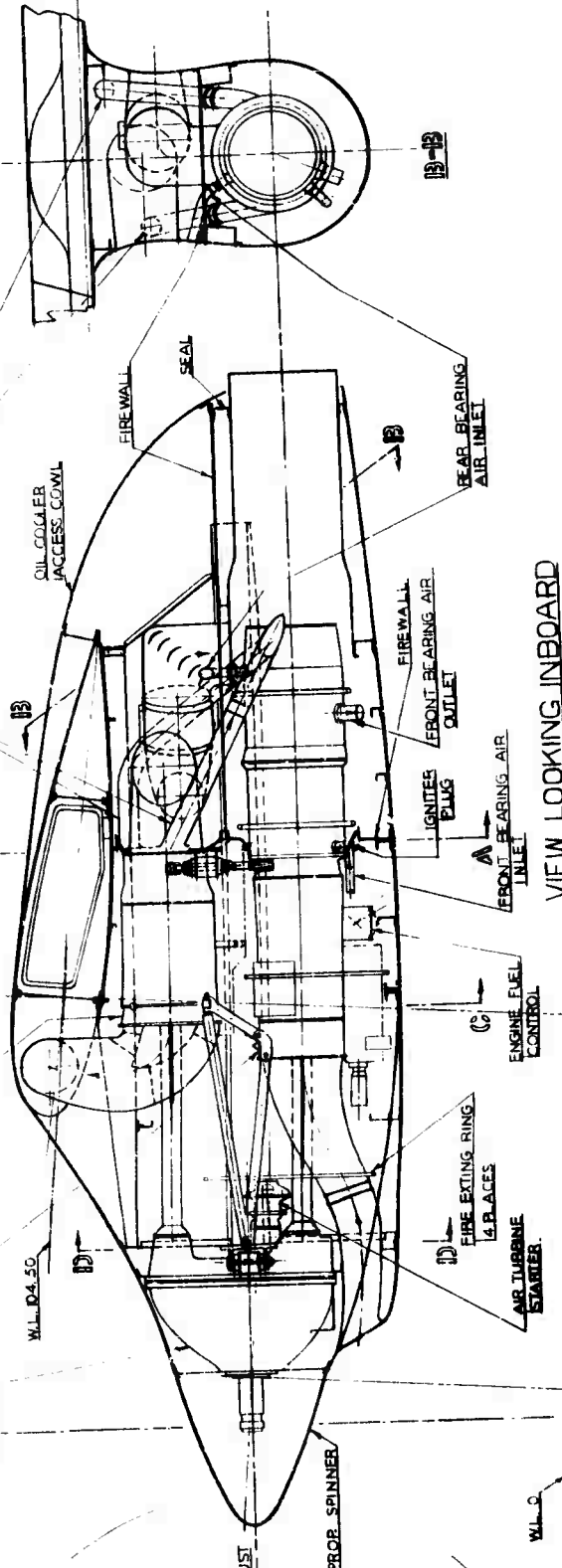
VERT. NACELLE

OIL TANK

COMPRESSED AIR DUCT TO ROTOR



LH NACELLE VIEW LOOKING AFT



NOTES:

1. FOR CONTINUATION OF COMPRESSED AIR DUCT SEE INBOARD PROFILE DWG. S-1518
2. ROTOR HUB ASSEMBLY DWG. S-2512

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POWER PLANT INSTALLATION  
S-15118

DATE 20 December 1940

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## 1. GENERAL POWER PLANT DESCRIPTION

1.1 General - The MAC Model 78 is provided with two gas turbine engines which drive either compressors for furnishing air to the pressure jet driven rotor or tractor propellers for high speed forward flight. A power unit nacelle is mounted on either side of the fuselage.

1.2 Engine - An Allison Model 301 gas turbine power section is mounted in each nacelle supplying shaft power to the propeller and compressor. A cooling air ejector is fitted to the engine exhaust pipe, drawing cooling air over the engine compressor section, combustion section, and oil radiator.

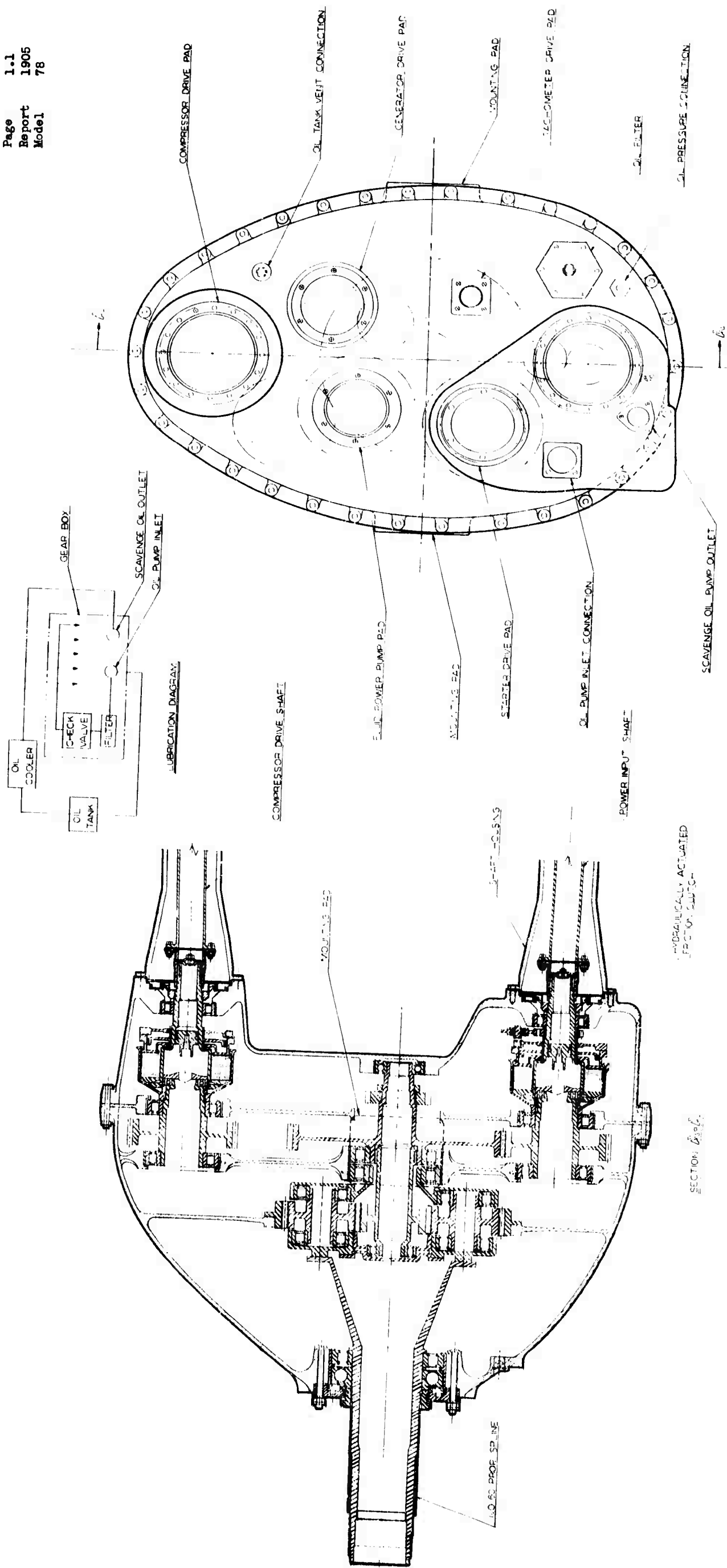
1.3 Pressure Jet Compressor - A Westinghouse 19XB axial flow compressor is employed to supply compressed air through a duct system to the rotor tip burners. This compressor is mounted directly above the engine with the compressor operated at engine speed by means of a drive shaft connected to the turbine shaft flange at the final discharge stage of the compressor.

1.4 Propeller - An Aeroproducts Model A632F propeller is mounted in the nose of each nacelle. During normal propeller operation, the propeller pitch is controlled by the engine control system. When the compressor is engaged to the engine, the propeller is held at the pitch resulting in minimum power absorption.

1.5 Gear Box - The engine drives the propeller and axial flow compressor by means of a modified Allison XT-38 gear box. The compressor shaft rotates in the opposite direction from the engine drive shaft at the same speed as the engine. Also included in the gear box section is one clutch permitting the engine to be disengaged from the gear box, and a second

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3. ENGINE RPM 14,300 PROP SHAFT RPM 1600  
2. OVERALL REDUCTION 1:785  
1. FIRST STAGE SPUR GEAR REDUCTION RATIO SAME AS THAT OF XT-38 PLANETARY GEARING PROP SHAFT 1 BEARINGS IDENTICAL WITH XT-38 GEAR BOX.

NOTES:

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**S-15117**

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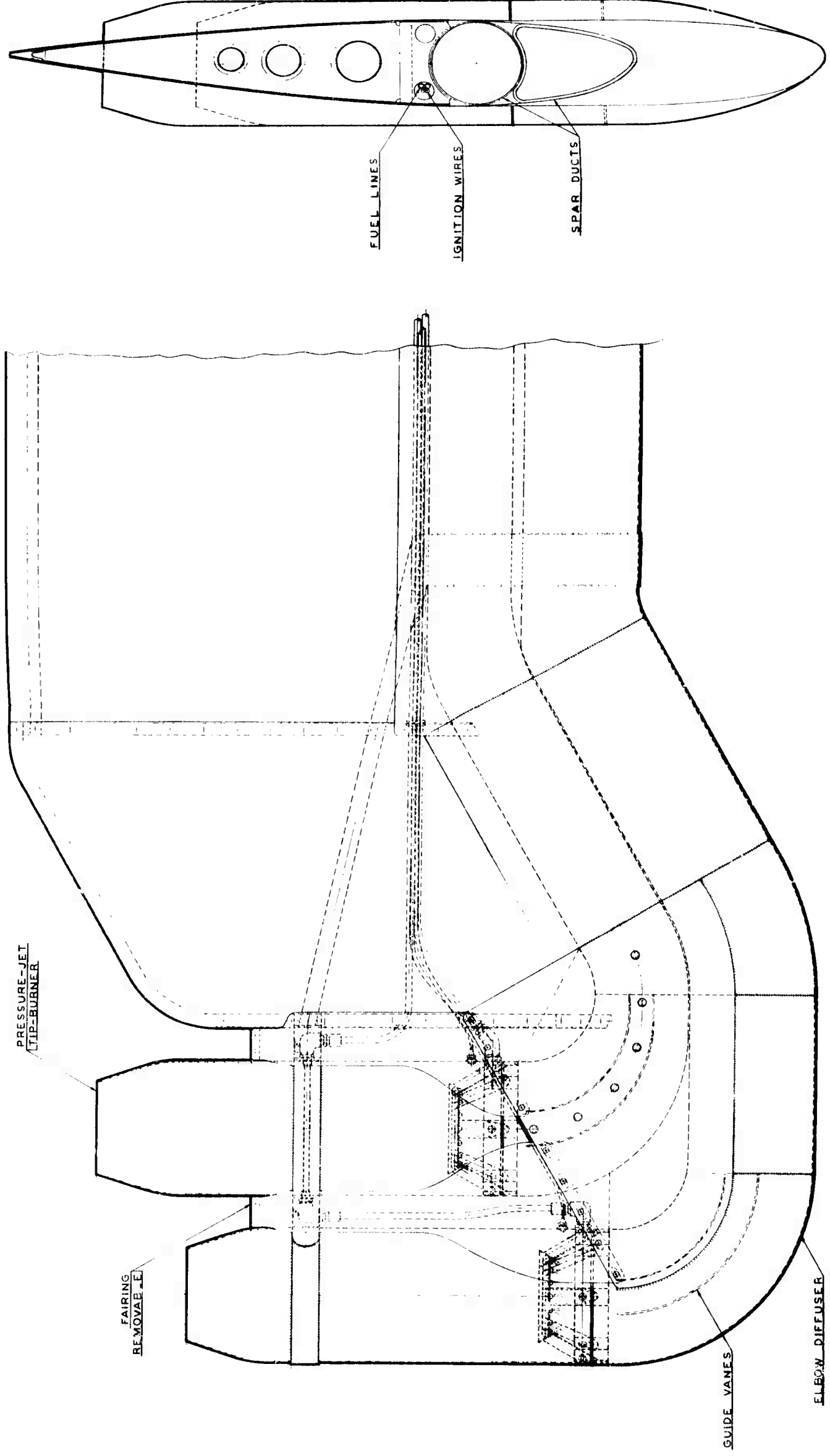
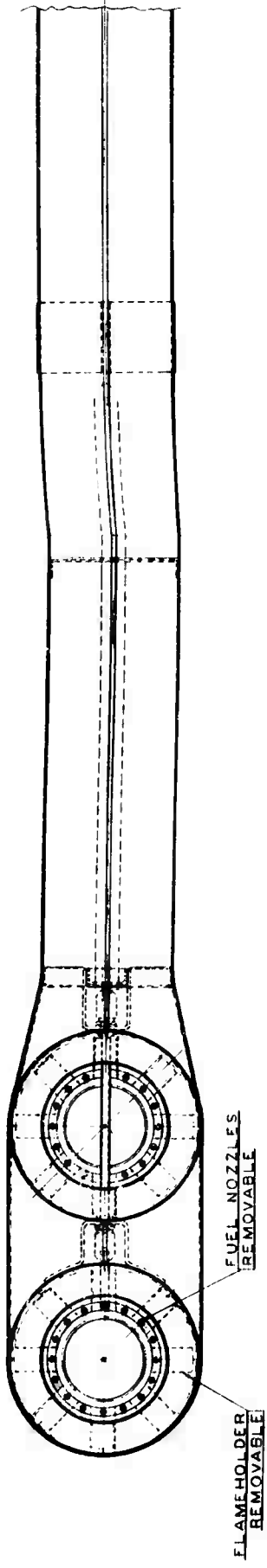
clutch which disengages the compressor. The rear face of the gear box provides the necessary accessory mounting pads and gear drives for the engine starter, generator, hydraulic pump, tachometer, and the connections for the gear box oil system and the propeller brake and governor controls. The starter gear train is connected to the engine drive shaft on the engine side of the clutch permitting the engine to be started while disengaged from the gear box.

1.6 Engine Starter - A single Airesearch air bleed gas turbine compressor is employed to drive a pneumatic starter mounted on each of the two engine gear boxes.

1.7 Rotor Tip Pressure Jets - Two burners are mounted at the tip of each of the three rotor blades. Compressed air supplied from the compressor is delivered to the tip burners through a duct system. Fuel is injected into the tip burners where combustion occurs, initiated by spark ignition.

1.8 Fuel System - The Model 78 fuel system is shown schematically in figure 1. This system was designed in accordance with specification SR-73D. Fuel cells totalling 627 gallons may be filled in a conventional method through the filler necks provided in each wing or by a single point pressure fueling fitting located on the left side of the fuselage. Refueling is made possible through the pressure fueling system by opening a shut-off valve located in the fuselage. This valve is normally closed to prevent inter-cell fuel flow through the pressure fueling system, while check valves prevent outboard fuel flow from the fuselage tank.

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Fuel is delivered to the engine fuel control by two submerged boost pumps located in the sump of the fuselage fuel cell. The fuel to each power section enters through a common selector valve. Tip burner fuel flow is controlled by a rotor driven governor-pump unit. Fuel inlet pressure for this unit is provided by the engine boost pump. When the tip burners are inoperative, fuel to the rotor system is shut off by a valve internal to the rotor governor. For single engine operation, a solenoid valve, located in the rotor hub shuts off the fuel to half of the tip burners.

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MODEL \_\_\_\_\_

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MODEL 75

1. The first step is to identify the problem or question that needs to be answered. This involves understanding the context and the specific requirements of the task.

During the period 1960-1962, the following information was obtained from the records of the Federal Bureau of Investigation, Department of Justice, regarding the activities of the following individuals:

[illegible]

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operating at the lower speed.

In order to obtain optimum pressure jet performance during single engine operation when the air flow is only one half of the normal flow, it is necessary to reduce the tip burner total exhaust nozzle area by 50 per cent. This is accomplished by closing the butterfly valves located in the aft duct in the root of each rotor blade, thus removing the three inner tip burners from the system. Actuation of this butterfly valve is automatically controlled as explained in section 2.3.

2.3 Pressure Jet Fuel Control - Fuel flow to the rotor tip burners is controlled by a rotor speed governor which meters fuel as necessary to maintain a constant selected rotor speed during all jet powered rotor operation. Thus as the rotor load changes, the pressure jet fuel flow is automatically adjusted to hold rotor speed, completely relieving the pilot of this duty. This type of rotor speed and tip jet fuel control has been performing satisfactorily in flight for a period of three years on the McDonnell XH-20 ram jet powered helicopter.

During periods of single engine operation, the flapper door of the air control valve (see section 2.2) actuates switches which closes a fuel shut-off solenoid valve located in the rotor hub. This solenoid valve shuts off the fuel flow to the three inner tip burners. Also connected to the manifold supplying fuel to the inner set of tip burners are actuating cylinders controlling the butterfly valves located in each rotor blade in the ducts supplying compressed air to the inner set of tip burners (see section 2.2). These cylinders are arranged so as to hold the butterfly valves open when fuel pressure exists in the manifold supplying fuel

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to the inner set of tip burners, and allows the butterfly valves to close when the solenoid valve shuts off the fuel in this manifold.

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MODEL 78

SYMBOLS

A	area	ft <sup>2</sup> unless otherwise noted
a	speed of sound	ft/sec
c <sub>p</sub>	specific heat at constant pressure	BTU/#°R
D	hydraulic diameter	ft
F	thrust	#
F <sub>0</sub>	compressibility factor	$1 - \frac{M^2}{4} + \frac{M^4}{40} + \frac{M^6}{1600} + \dots$
f	wall friction factor	
f/a	fuel-air ratio	
g	acceleration of gravity	32.2 ft/sec <sup>2</sup>
H	total head	#/ft <sup>2</sup>
HP	horsepower	
H <sub>2</sub> /E <sub>0</sub>	total pressure recovery	
h <sub>v</sub>	lower heating value	18,000 BTU/# for gasoline
J	mechanical equivalent of heat	778 ft·# / BTU
l	length	ft
M	Mach number	
m	mass rate of flow	slugs/sec
P	absolute static pressure	#/ft <sup>2</sup> unless otherwise noted
P <sub>T</sub>	absolute total pressure	#/ft <sup>2</sup> unless otherwise noted
Q	volume rate of flow	ft <sup>3</sup> /sec unless otherwise noted
q	dynamic pressure	#/ft <sup>2</sup> unless otherwise noted
q <sub>0</sub>	impact pressure qF <sub>0</sub>	#/ft <sup>2</sup> unless otherwise noted
R	gas constant	ft·# / #°R

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RN	Reynolds number	
SHP	shaft horsepower	
T	static temperature	OR
T <sub>T</sub>	total temperature	OR
V	velocity	ft/sec
V <sub>T</sub>	tangential velocity	ft/sec
v	specific volume	ft <sup>3</sup> /#
W	weight rate of flow	#/sec
α	angle of attack	degrees
γ	ratio of specific heats	
Δ	used to represent a change in another quantity	
δ	P <sub>T2</sub> /14.7 when P <sub>T2</sub> is expressed in PSIA	
θ	T <sub>T2</sub> /18.4	
η	efficiency	
ρ	mass density	slugs/ft <sup>3</sup>
ω	rotational velocity	radians/sec

#### Subscripts

0	free stream	i	inlet
1, 2, etc.	station numbers	j	jet
a	air	ot	total loss
b	burning	p	primary
c	compressor	s	secondary
f	fuel	TR	temperature rise
g	gas - products of combustion		

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MODEL 78

### 3. ANALYSIS OF POWER PLANT DUCTS

3.1 General - The analysis of the Model 78 duct system is presented in accordance with the U. S. Navy Aeronautical Specification **NAVAR 88-150**, "Specification for the Calculation of Duct losses, Turbo-Jet and Gas Turbine Engines".

This section includes the analysis of the inlet ducts supplying air to the two Allison Model 501 power sections, the inlet ducts supplying air to the two Westinghouse 19XB compressors, and the engine exhaust system including the cooling ejector.

3.2 Description of Ducts - The engine inlet duct is approximately five feet long with a well-rounded lip. The duct increases in cross-sectional area from 160 square inches at the inlet to 186 square inches at the engine face. The annular flow area at the engine compressor face is 160 square inches.

The 19XB compressor inlet duct is about three feet long and has a well-rounded lip. The duct increases in cross-sectional area from 140 square inches to 160 square inches at a station near the compressor face. The annular area at the compressor face is 138 square inches.

The engine exhaust tailpipe is shrouded with an ejector which is split by radial dividers into three parts drawing cooling air through the forward engine compressor compartment, the engine combustion chamber compartment, and the oil cooler.

Figure 3 presents a schematic of the ducting arrangement and cross-sectional areas.

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3.3 Results P Arrayed - The total pressure receiver, for the engine and the compressor inlets is shown in figure 4. It is said in a series of wind tunnel and flight tests conducted by MAC on comparable installations (references 1 and 2).

Tests of similar insulators indicate that the oil and compressor inlet ducts should have a total mass recovery of 92% under sea level static conditions without benefit of preheated ram.

Tables 1 and 2 present the detailed analysis of the data for the center of the 1.0 and 3.0 mm. diam. beams. The analysis of the 3.0 mm. beam area, equivalent to the total area of diffusion, is shown in Table 3, and the total velocity of mass axial transport and pressure gradient is shown in Table 4.8 for the 1.0 and 3.0 mm. diam. beams.

the island (1 mile by 1/2 mile) is the only virgin forest left in the area. It is a small island and is not a part of the main island. It is a small island and is not a part of the main island. It is a small island and is not a part of the main island.

Data presented in reference 1 shows that a secondary air flow of 0.3 l/s/cm<sup>2</sup> can be induced by an air flow of 9.1 l/s<sup>2</sup> (static sea level, normal power). This ejector is shown as indicated in Figure 1 for the power plant cooling requirements. Tests of a jet engine afterburner or ejector and a jet ejector in line have shown that low cooling losses may be expected from this design. Cross section area, equivalent conical angle of diffusion and exhaust Mach number in axial flow for the ejector ducts are presented in Figures 11 and 12.

The seal, oil, and air cooled air duct system in the compressor  
to monitor the temperature, and the temperature of the oil, included  
in the pressure jet oil pump and oil.

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MODEL 78

#### 4. ALLISON MODEL 501 LOWER SECTION PERFORMANCE

4.1 Engine Performance - The Allison Model 501 gas turbine engine performance was evaluated by the method presented in the engine specification (reference 3). Reference 3 represents the most recent Navy approved specification for the power section performance. Inlet duct total pressure recovery was based upon the duct analysis presented in section 3. Values of  $H_2/H_0$  from figure 4 were used for the engine performance calculations.

During powered-rotor flight with the propeller in the pitch resulting in minimum power absorption it is assumed that the propeller absorbs 6% of the available engine shaft horsepower. Accordingly the power available to drive the compressor, as presented in figure 13 has been reduced by 6%.

As noted in section 2.1 the engine will be operated at constant normal speed (14,000 rpm) during all periods that it is engaged to the compressor. This necessitates, under certain conditions, that the turbine inlet temperature exceed the normal rated temperature, but under no conditions will it exceed the allowable military temperature. In view of the allowable operating time of 30 minutes at military power this type of operation is considered satisfactory especially as it is accomplished at a reduced engine speed. Figure 13 presents a plot of horsepower available and horsepower required versus altitude for the most critical conditions at 14,000 rpm and a turbine-inlet temperature of 1935°R.

Figures 14 through 22 present shaft horsepower, net jet thrust, and fuel flow for propeller operation of each engine in the anticipated operating ranges of flight velocity and altitude. Performance was determined in accordance with the engine specification (reference 3) using the estimated inlet total pressure recovery presented in figure 4.

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MODEL \_\_\_\_\_

207-11330- J-1 WILSON, ALFRED E. NOV

1. Heat Exchanger Performance - The heat exchanger data available from the test, as shown in Figure 1, are plotted in the text of the report. In order to determine compressor performance it was necessary to construct a new horsepower required versus pressure ratio curve based upon the static temperature rise efficiency data shown in Figure 2. The question:

$$HP = \frac{1}{\eta_{TR}} \frac{RT}{550} (\gamma/\delta-1) \left[ \left( P_3/P_{T2} \right)^{\frac{\delta-1}{\delta}} - 1 \right] W_a$$

was employed for this purpose. A resultant curve ( $\sigma_1$  vs  $\sigma_2$ ) was checked by Westinghouse and found to be accurate throughout. The stress ratio for prior annealing is zero. Figures 10 and 11 are as presented in reference 6.

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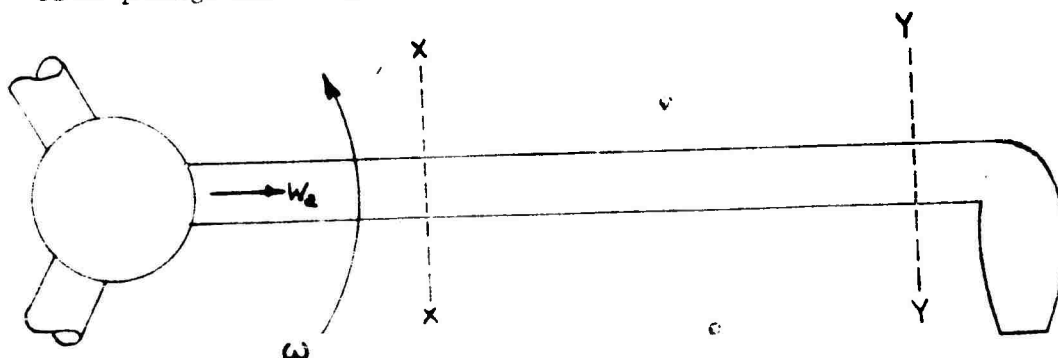
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Considering a rotor blade as shown below with air flowing through the blade passage and being burned in the tip burner, the power required to



accelerate the air from the tangential velocity at station X to the tangential velocity at station Y is:

$$\text{POWER} = \frac{W_a (V_{Ty}^2 - V_{Tx}^2)}{2g}$$

2

also, the power required for isentropic compression of the air from station X to station Y is: (reference 7).

$$\text{POWER} = J C_p T_{Tx} \left[ \left( \frac{P_{Ty}}{P_{Tx}} \right)^{\frac{\gamma-1}{\gamma}} - 1 \right] W_a$$

3

Equating equation 2 and equation 3

$$\frac{W_a (V_{Ty}^2 - V_{Tx}^2)}{2g} = J C_p T_{Tx} \left[ \left( \frac{P_{Ty}}{P_{Tx}} \right)^{\frac{\gamma-1}{\gamma}} - 1 \right] W_a$$

$$\left( \frac{P_{Ty}}{P_{Tx}} \right)^{\frac{\gamma-1}{\gamma}} = 1 + \frac{(V_{Ty}^2 - V_{Tx}^2)}{2g J C_p T_{Tx}}$$

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$$\frac{P_{T_y}}{P_{T_x}} = \left[ 1 + \frac{(V_{T_y}^2 - V_{T_x}^2)}{2gJ C_p T_{T_x}} \right]^{\frac{\gamma}{\gamma-1}}$$

Since  $J C_p = R(\gamma/\gamma-1)$ , the equation for the pressure ratio developed from station X to station Y may now be written:

$$\frac{P_{T_y}}{P_{T_x}} = \left[ 1 + \frac{(V_{T_y}^2 - V_{T_x}^2)}{2gR T_{T_x} (\gamma/\gamma-1)} \right]^{\frac{\gamma}{\gamma-1}}$$

4

2.2.2 Pressure Losses - To determine the pressure losses in the system due to the flow of air through the ductwork, the tip burner, a shock-wave analysis was made for each calculated diameter per passage. The method and data of reference 2 were used to evaluate the pressure losses. Pumpin' gains due to rotor rotation using the method presented in reference 2.

2.2.1 and momentum pressure loss (evaluated with reference 2) were also taken into consideration. After the pressure available in the tip burner for combustion was determined, the tip burner performance was evaluated in the manner discussed below.

2.2.3 Design Condition - A design condition was first selected which was the basis for determining the nozzle exit area of the tip burner. The design condition chosen was a corrected pressure ratio ( $P/P_0$ ) of 10,000

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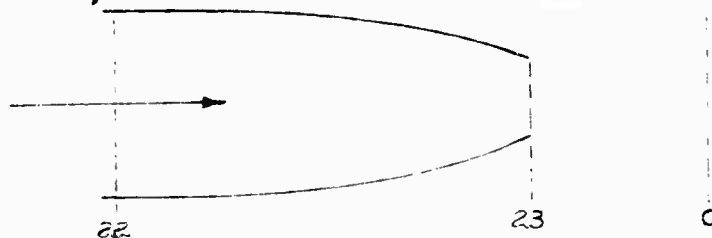
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MODEL

operation at a compressor pressure ratio  $(P_{23}/P_{22})$  of 2.0. This is a safe condition. Since the compressor stall indicates the pressure is insufficient to maintain a critical velocity through the nozzle exit area, the area was selected on this basis.



$$P = \rho RT$$

$$\rho = \frac{P}{RT} = \frac{W}{A V}$$

$$\frac{W}{A} \frac{RT}{P} = A V$$

For critical velocity at nozzle exit

$$V_{23} = \sqrt{\gamma RT_{23}}$$

6

From critical velocity and equation 6

$$\frac{P_{23}}{P_{22}} = \left( \frac{2}{\gamma + 1} \right)^{\frac{\gamma}{\gamma - 1}}$$

$$P_{23} = P_{22} \left( \frac{2}{\gamma + 1} \right)^{\frac{\gamma}{\gamma - 1}}$$

7

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Substituting equations 6 and 7 into equation 5 the following expression is obtained for the nozzle area.

$$\frac{W_g RT_{23}}{P_{22} \left(\frac{2}{\gamma+1}\right)^{\frac{\gamma}{\gamma-1}}} = A_{23} \sqrt{\gamma g RT_{23}} \quad 8$$

$$A_{23} = \frac{W_g RT_{23}}{P_{22} \left(\frac{2}{\gamma+1}\right)^{\frac{\gamma}{\gamma-1}} \sqrt{\gamma g RT_{23}}} \quad 9$$

In view of the fact that burning efficiency is low, it was assumed that no performance loss due to incomplete burning would be obtained in the combustor chamber. Test data on a similar combustor reported in reference 9 indicate burner to combustor efficiency of this order and higher. Therefore the nozzle exit area was determined for the maximum air flow condition at a total temperature of 4,000 R. Internal boundary layer air cooling was neglected. The combustor pressure was assumed to be 100 psia.

The relation between total temperature and total area temperature at Mach 1 is:

$$T_T = T \left(1 + \frac{\gamma-1}{2} M^2\right) \quad 10$$

SINCE  $M = 1.0$

$$T_{23} = 4000 \left(\frac{2}{\gamma+1}\right)$$

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From equation 9, the nozzle area now becomes:

$$A_{23} = \frac{W_g R (4000) \left(\frac{2}{\gamma+1}\right)}{P_{T22} \left(\frac{2}{\gamma+1}\right)^{\frac{\gamma}{\gamma-1}} \sqrt{\gamma R (4000) \left(\frac{2}{\gamma+1}\right)}} \quad 11$$

$$\frac{f}{a} = \frac{C_p \Delta T}{h_v \eta_B}$$

Assume a burning efficiency of 90% and a lower heating value of gasoline  
 = 18,000 BTU/#.

$$\frac{f}{a} = \frac{C_p \Delta T}{16,200} \quad 12$$

Since the values of  $\gamma$  and  $C_p$  are dependent upon temperature and fuel-air ratio it is now necessary to make successive approximations for fuel-air ratio and cycle through equations 11 and 12 until both equations are satisfied.

$P_{T22}$  is determined from the duct analysis as described in section 5.2.2 and a trial and error solution is readily made since there is only one fuel-air ratio that will give a total temperature of 4000°R.

On this basis a total nozzle area of 0.938 square feet was determined. This total nozzle area is divided by the number of tip burners to obtain the area per burner. For the six burners the effective nozzle exit area per burner is 22.5 in.<sup>2</sup> or 5.35 in. in diameter.

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5.2.4 Performance Calculations - For other corrected compressor speeds and pressure ratios the combustion temperature may be determined since the area has been determined. Transposing equation 8:

$$\frac{T_{23}}{\sqrt{T_{23}}} = \frac{A_{23} \sqrt{\gamma g R} P_{T22} \left(\frac{\gamma}{\gamma+1}\right)^{\frac{\gamma}{\gamma-1}}}{W_g R}$$

$$T_{23} \left[ \frac{A_{23} \sqrt{\gamma g R} P_{T22} \left(\frac{\gamma}{\gamma+1}\right)^{\frac{\gamma}{\gamma-1}}}{W_g R} \right]^2$$

13

Using equation 12 and 13 and determining  $P_{T22}$  from the rotor duct analysis  $T_{23}$  may be determined for any compressor condition. Although equation 13 applies only when  $P_{T23}$  is above the critical pressure ratio, this condition covers most of the system operating range.

Fuel flow is determined by:

$$W_f = f/a W_a = \frac{C_p \Delta T}{16,200} W_a$$

14

Total temperature at the nozzle exit for sonic velocity is

$$T_{T23} = T_{23} \left(\frac{\gamma+1}{2}\right)$$

15

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The jet velocity is obtained from equation 6. Jet thrust is obtained as follows:

$$F_J = \eta_N \frac{W_a}{g} (V_{23} - V_T) + (P_{23} - P_0) A_{23} \quad 16$$

$\eta_N$  = nozzle efficiency (assumed .95)

A sample calculation is presented in table 7. The first section of the calculation is the rotor system duct analysis to determine the pressure available for burning. The latter part of the calculation presents the tip burner performance for the available pressure. It was assumed that the division of air flow through the two flow passages in the rotor blade (station 17 through 23) was equal, even though the cross-sectional areas are slightly different (22.1 and 24.3 square inches per blade). The actual division of the air flow will be governed by the back pressure of each tip burner due to burning. The total temperature, fuel flow, and jet thrust were calculated for each of the two tip burners and the average total temperature was used for the total temperature of the gas.

Curves of corrected performance over a range of thrust values are presented in figures 26 and 27. Corrected values are presented in the general curves since they are independent of the compressor inlet conditions. The correction factors used are the standard factors used in correction of jet engine performance. Reference 10 presents the derivation of these factors. These factors are:

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MODEL 78

$$\delta = P_{T_2}/14.7 \text{ where } P_{T_2} \text{ is expressed in PSIA}$$

$$\theta = T_{T_2}/518.4 \text{ where } T_{T_2} \text{ is expressed in degrees Rankine}$$

Several numerical examples were checked for various altitudes within the performance range of Model 78. Agreement of corrected performance with that determined by using the actual temperature and pressure was obtained.

Actual tip burner thrust and overall fuel consumption are presented in figures 28 through 31 for the anticipated operating range of the tip burners.

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MODEL 78

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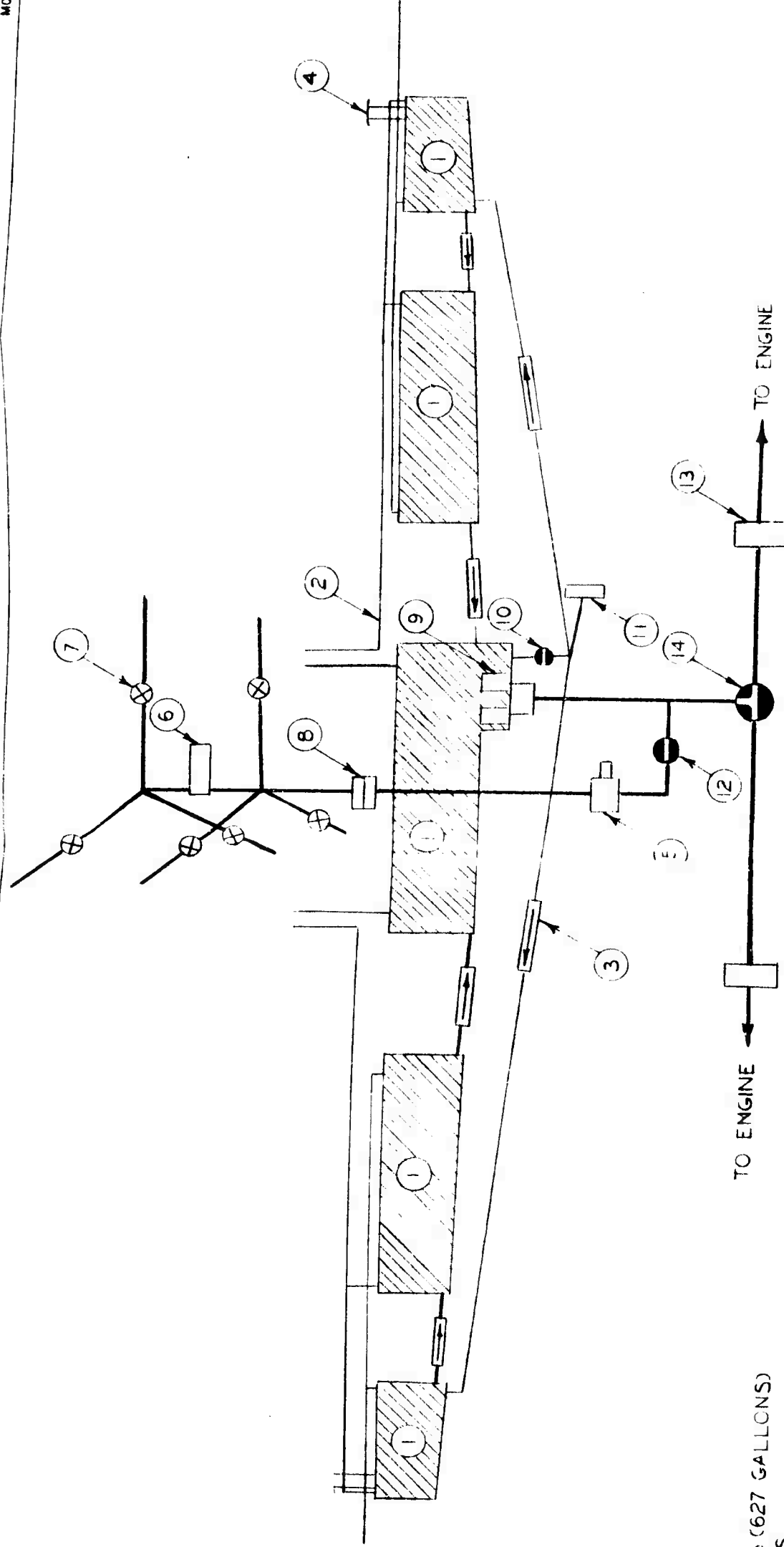
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MODEL 78



- 1 FUEL CELLS (627 GALLONS)
- 2 VENT LINES
- 3 CHECK VALVES
- 4 FILLER NECK
- 5 ROTOR DRIVEN PUMP-GOVERNOR UNIT
- 6 SHUT OFF VALVE (FOR SINGLE ENGINE HELICOPTER FLIGHT ONLY)
- 7 ORIFICE
- 8 RECTARY JOINT
- 9 BOOST PUMP
- 10 SHUT OFF VALVE - NORMALLY CLOSED
- 11 PRESSURE FUELING & DEFUELING FITTING
- 12 SHUT OFF VALVE
- 13 FILTER
- 14 SELECTOR VALVE-LEFT-RIGHT-BOTH-OFF

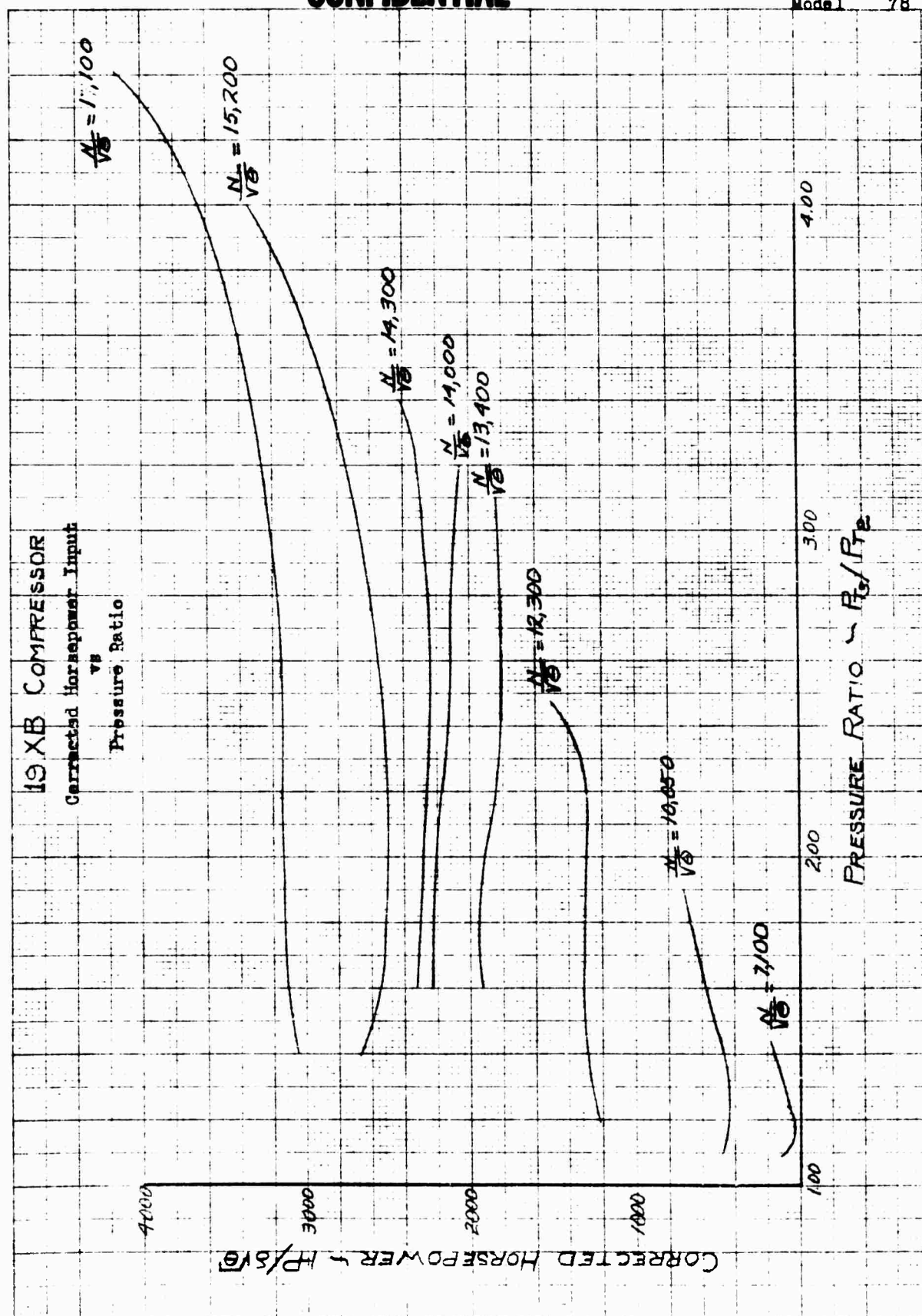
MODEL 78  
FUEL SYSTEM SCHEMATIC

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FIG 1



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FIG. 2

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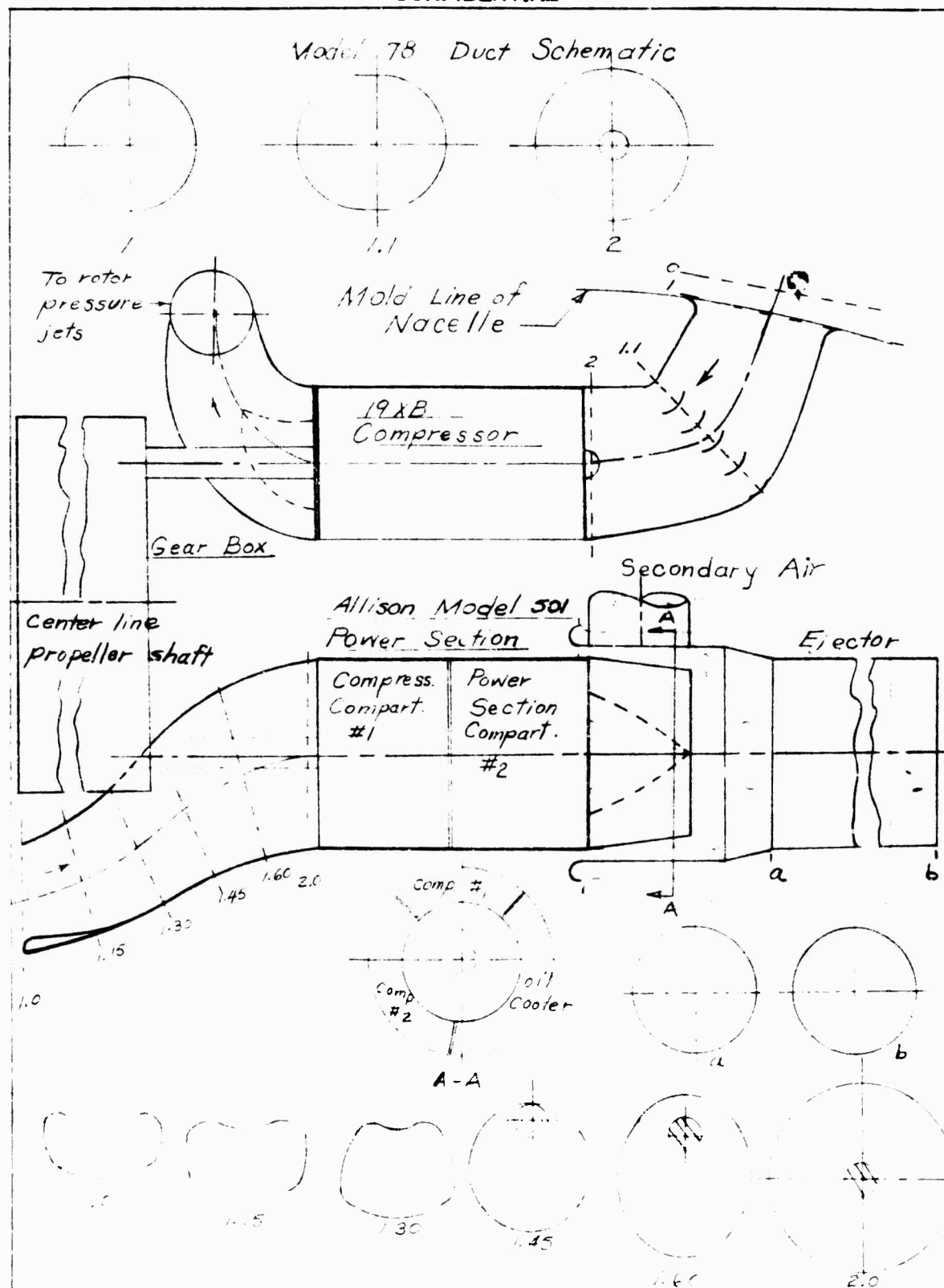
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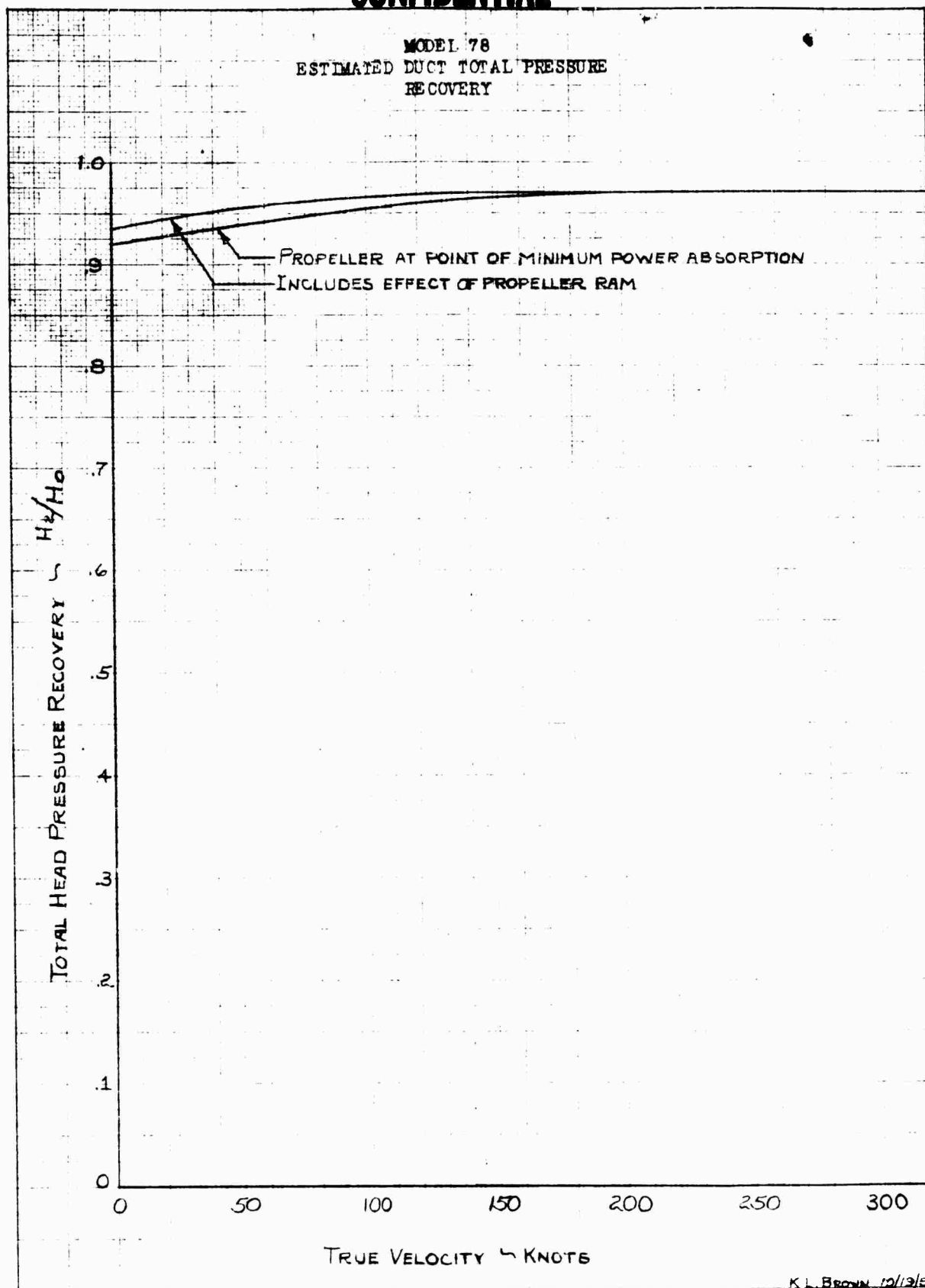
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FIG. 3

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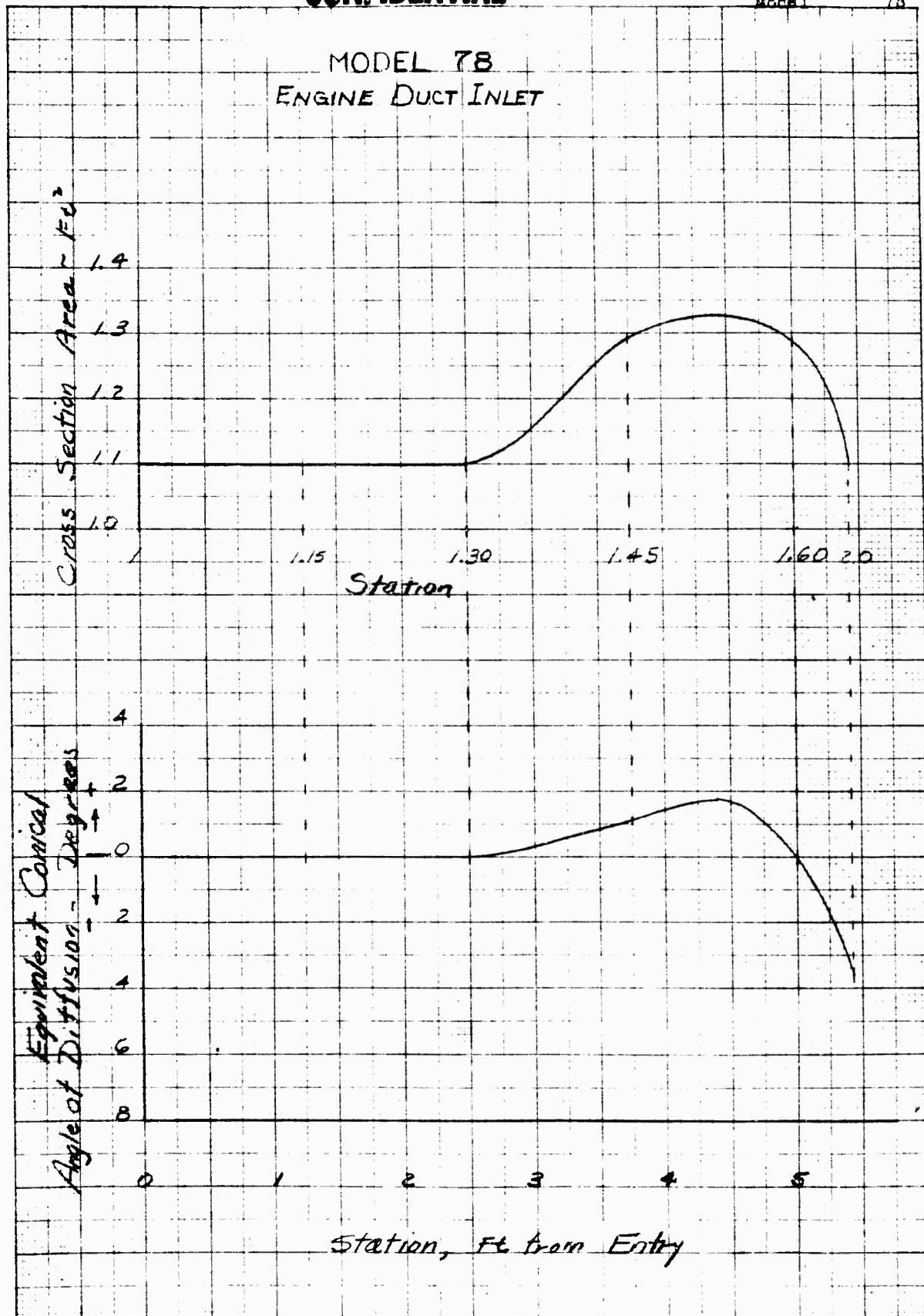
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FIG. 4

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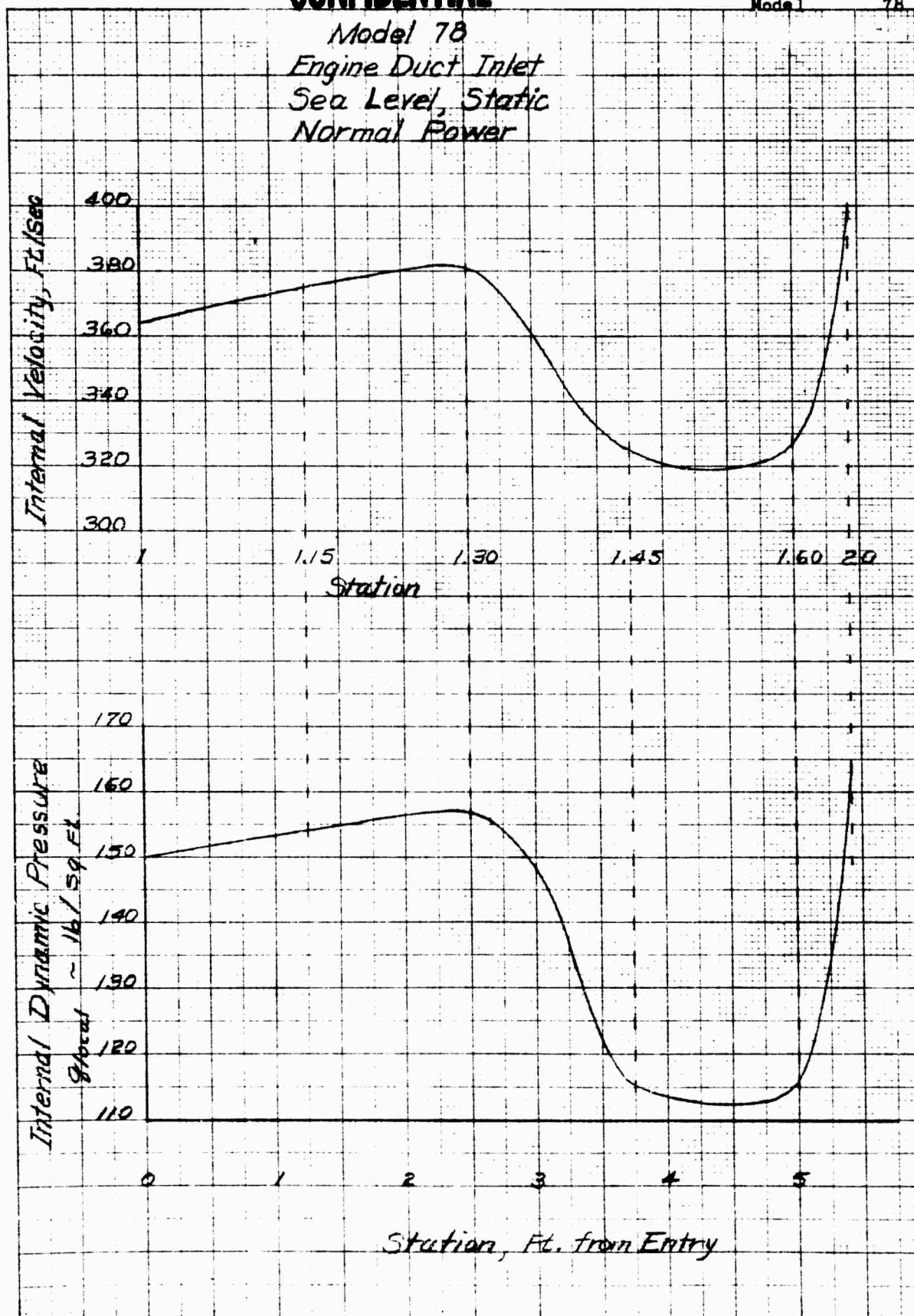
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FIG. 5

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FIG 6

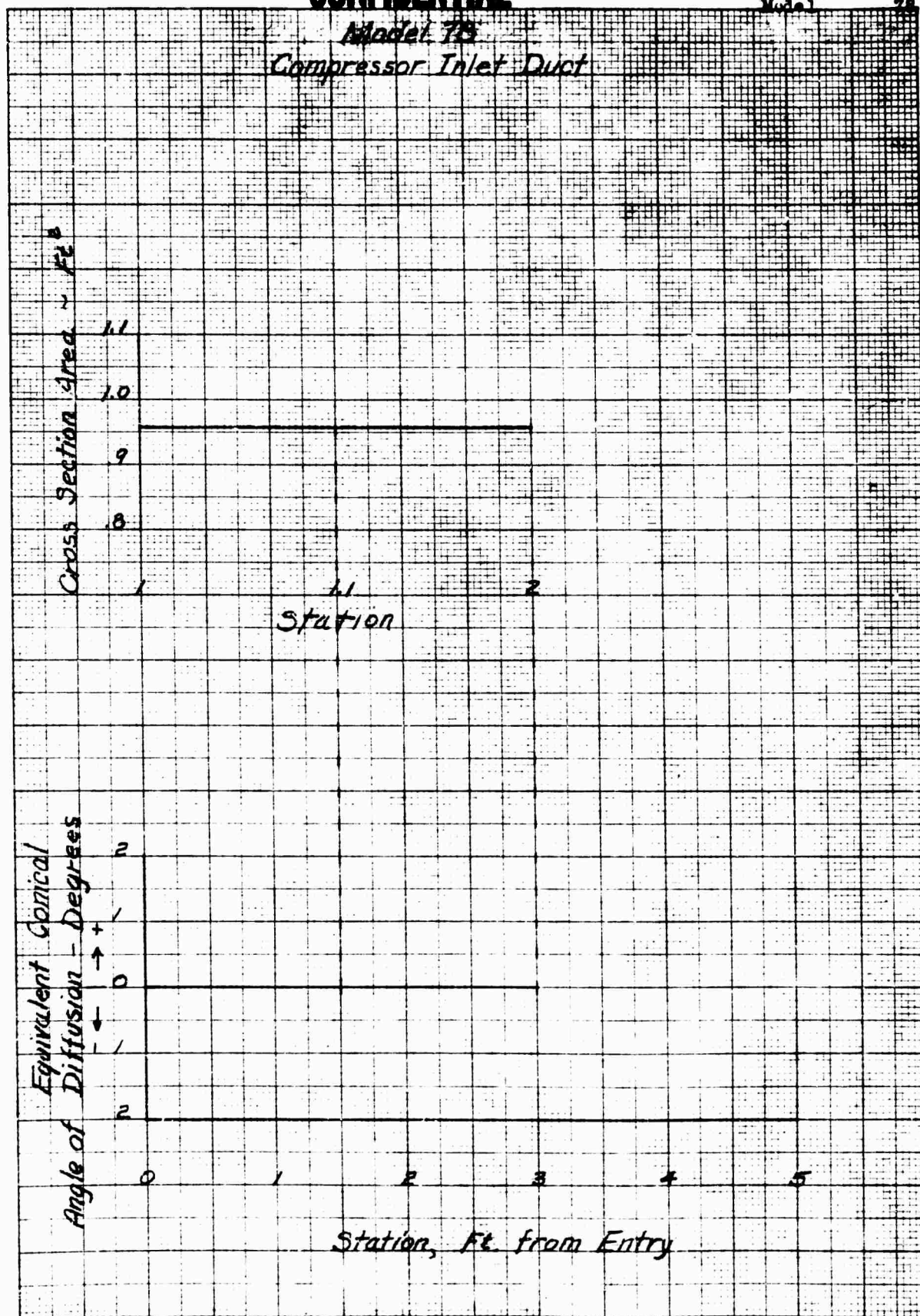
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*Model 7B*  
*Compressor Inlet Duct*



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FIG. 7

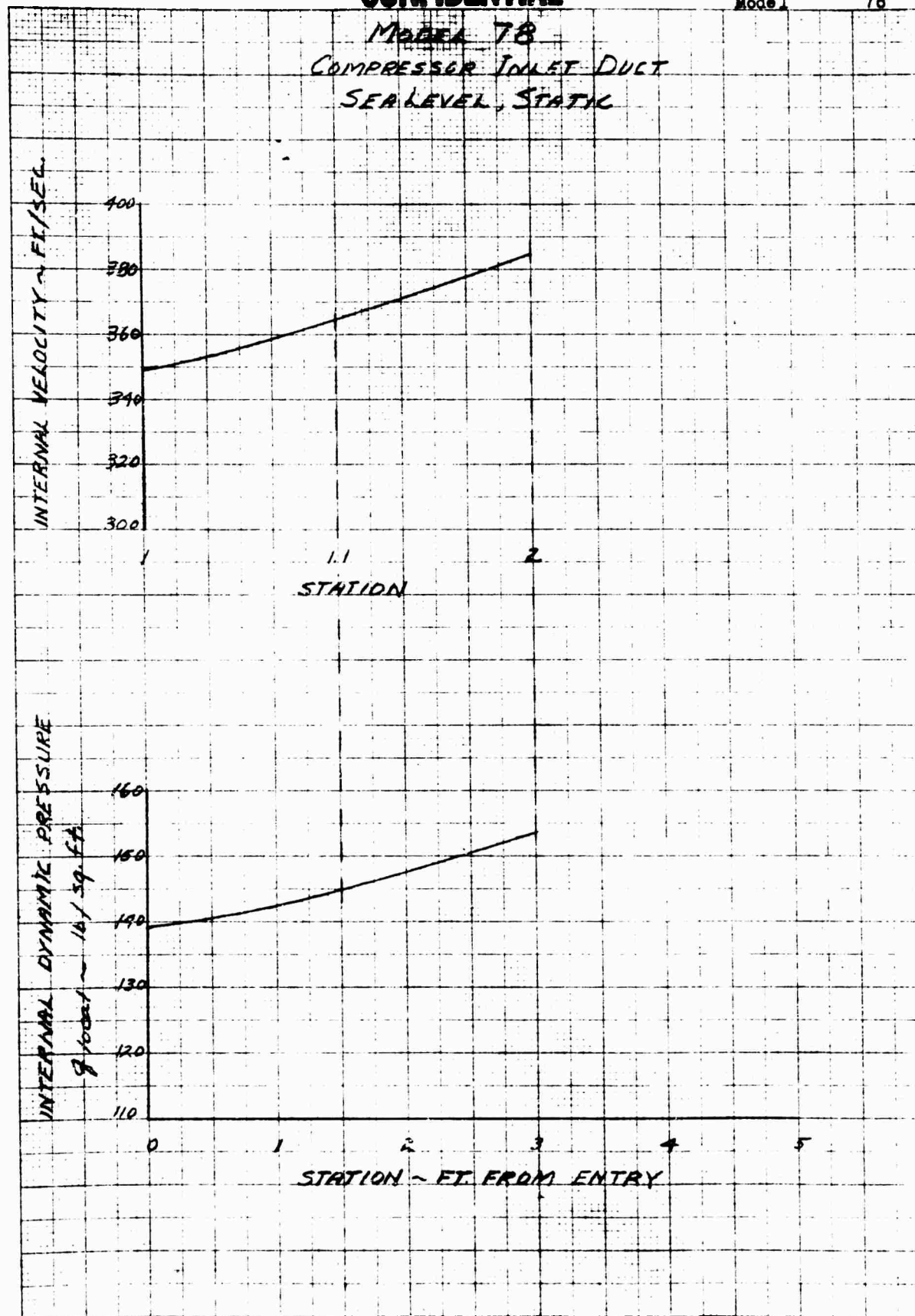


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MODEL 78  
COMPRESSOR INLET DUCT  
SEA LEVEL, STATIC



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FIG. 9

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MODEL 73

Conditions: Sea Level, Static, Normal Power (14,000 RPM)

$$P = 575 \text{ #}, \quad P_{22} = 11 \text{ #/sq in}, \quad \gamma = 1.4$$

$$W_0 = 28.45 \text{ #/sec}, \quad W_f = 1473 \text{ #/hr} = 4.09 \text{ #/sec}$$

$$W_0/W_f = 0.0144, \quad \sigma = 1.353, \quad T_f = 1295^\circ \text{R}$$

$$F = \gamma \frac{W_0}{A_0} V_0 - W_f V_0 = 95 \times \frac{28.45}{22.2} \times V_0 - 0 = 575$$

$$V_0 = \frac{575 \times 22.2}{95} = 676 \text{ Ft/sec}$$

From the ideal jet velocity equation, solve for the nozzle pressure ratio.

$$V = \sqrt{2 \gamma R \frac{\sigma}{\gamma} \left[ 1 - \left( \frac{P_0}{P_f} \right)^{\frac{\gamma-1}{\gamma}} \right]}$$

$$676 = \frac{2 \times 32.2 \times 53.3 \times 1.353 \times 1295}{1.353 - 1} \left[ 1 - \left( \frac{P_0}{P_f} \right)^{\frac{1.4-1}{1.4}} \right]$$

$$\frac{P_0}{P_f} = (0.732)^{3.875} = .90$$

$$P_f/P_0 = \frac{1}{.90} = 1.12$$

The exhaust nozzle pressure ratio is 1.12.

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FIG. 9



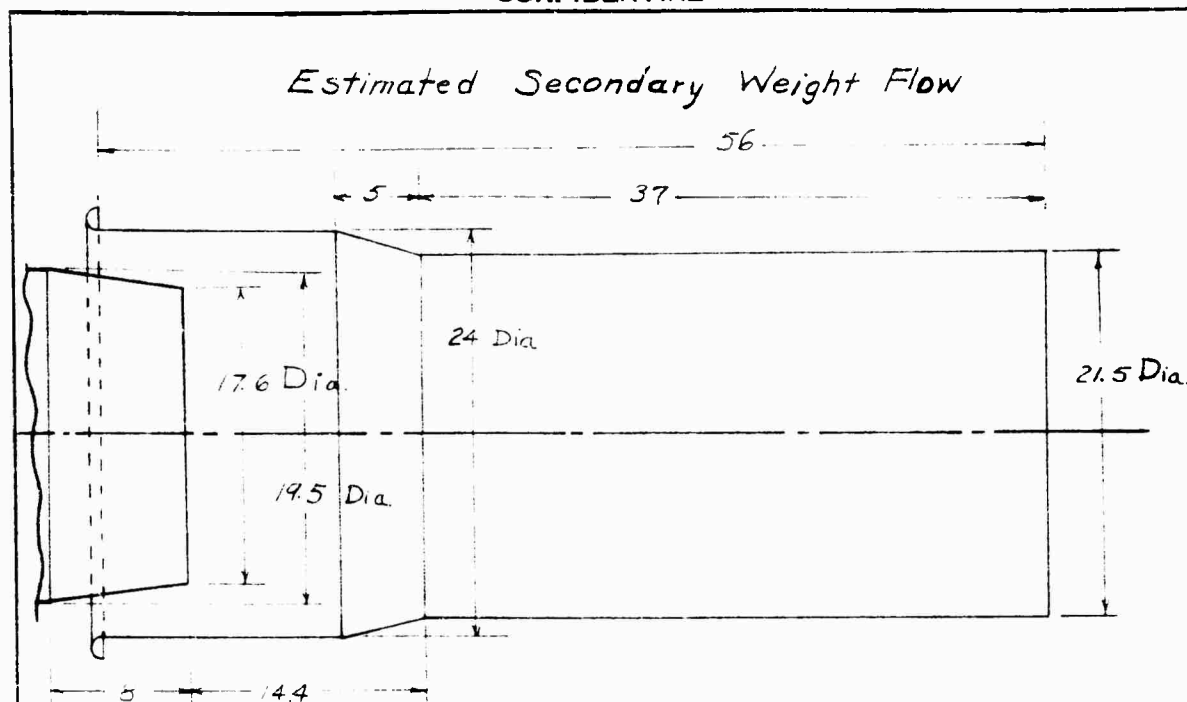
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The corrected weight flow ratio from reference 4.

$$\frac{W_S}{W_P} \sqrt{\frac{T_S}{T_P}} = .14$$

$$T_S = 520^\circ R$$

$$T_P = 1295^\circ R$$

$$W_S/W_P = .14 \sqrt{\frac{T_P}{T_S}} = .14 \sqrt{\frac{1295}{520}} = .222$$

$$W_P = 28.45 \text{ lb/sec}$$

$$W_S = .222 \times 28.45 = 6.28 \text{ lb/sec}$$

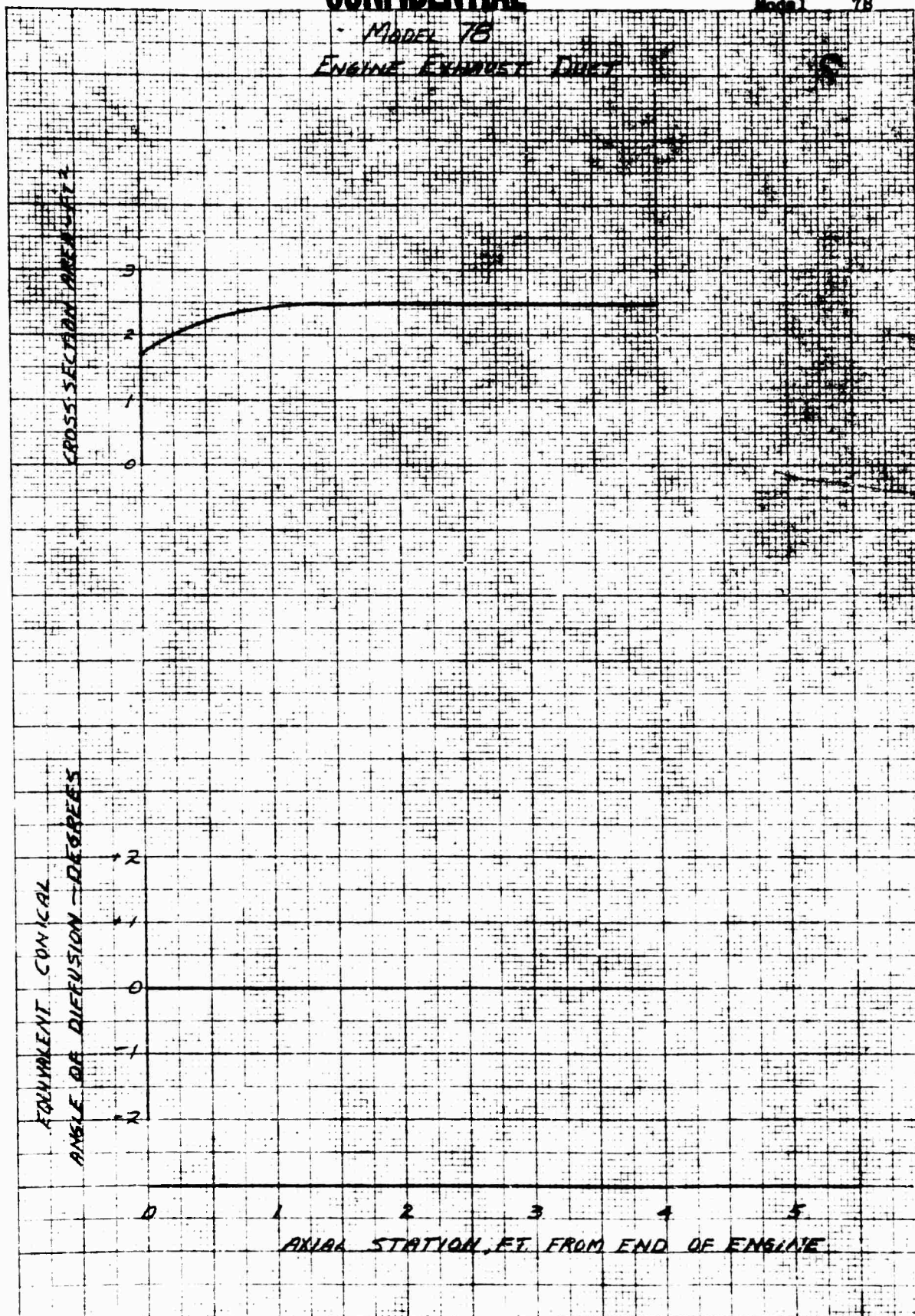
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FIG. 10

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FIG. 11

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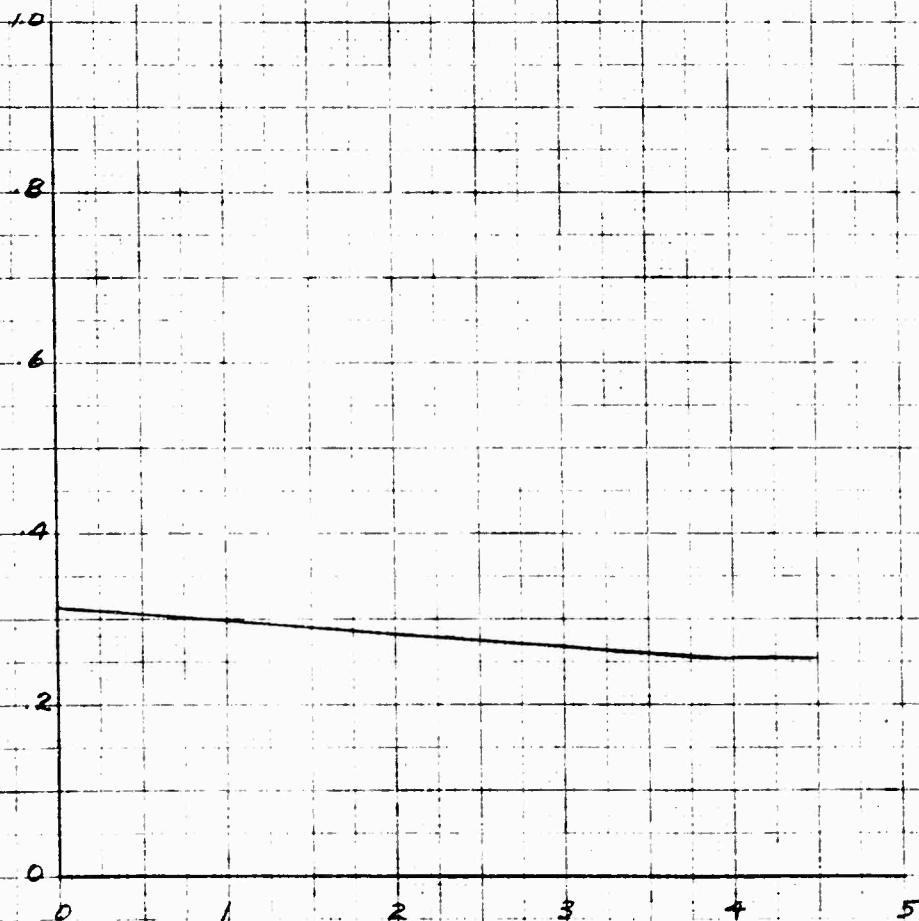
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MODEL 78  
ENGINE EXHAUST DUCT  
SEA LEVEL, STATIC, NORMAL POWER

EXHAUST DUCT MACH NUMBER

AXIAL STATION - FT. FROM END OF ENGINE



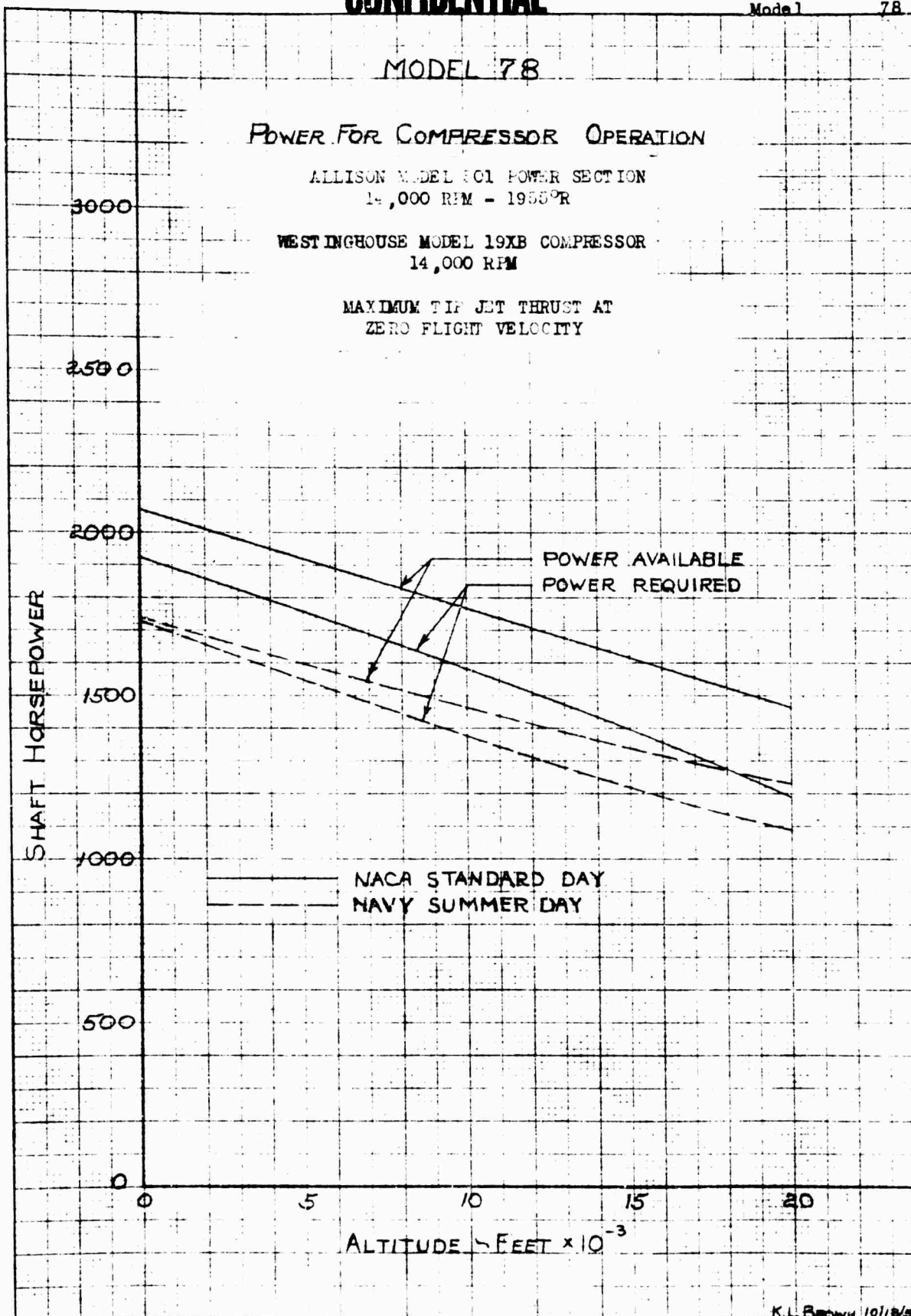
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FIG 12

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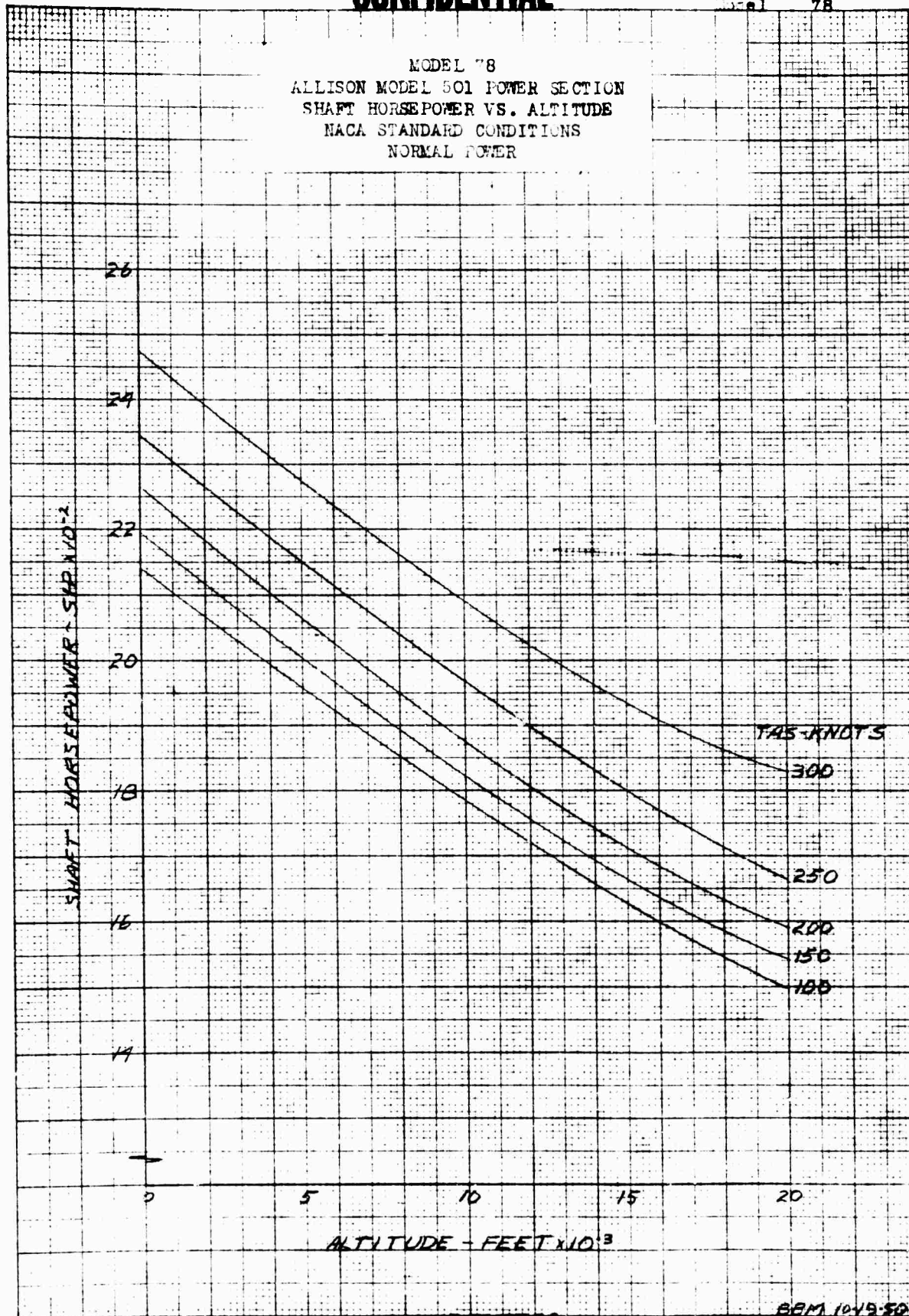
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FIG 13

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REUPPEL & BISSER CO., N. Y. NO. 359-11  
18 X 18 in the 1/4 inch 5th line extended  
U.S. N. O. S. 4

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BEM 1049-50

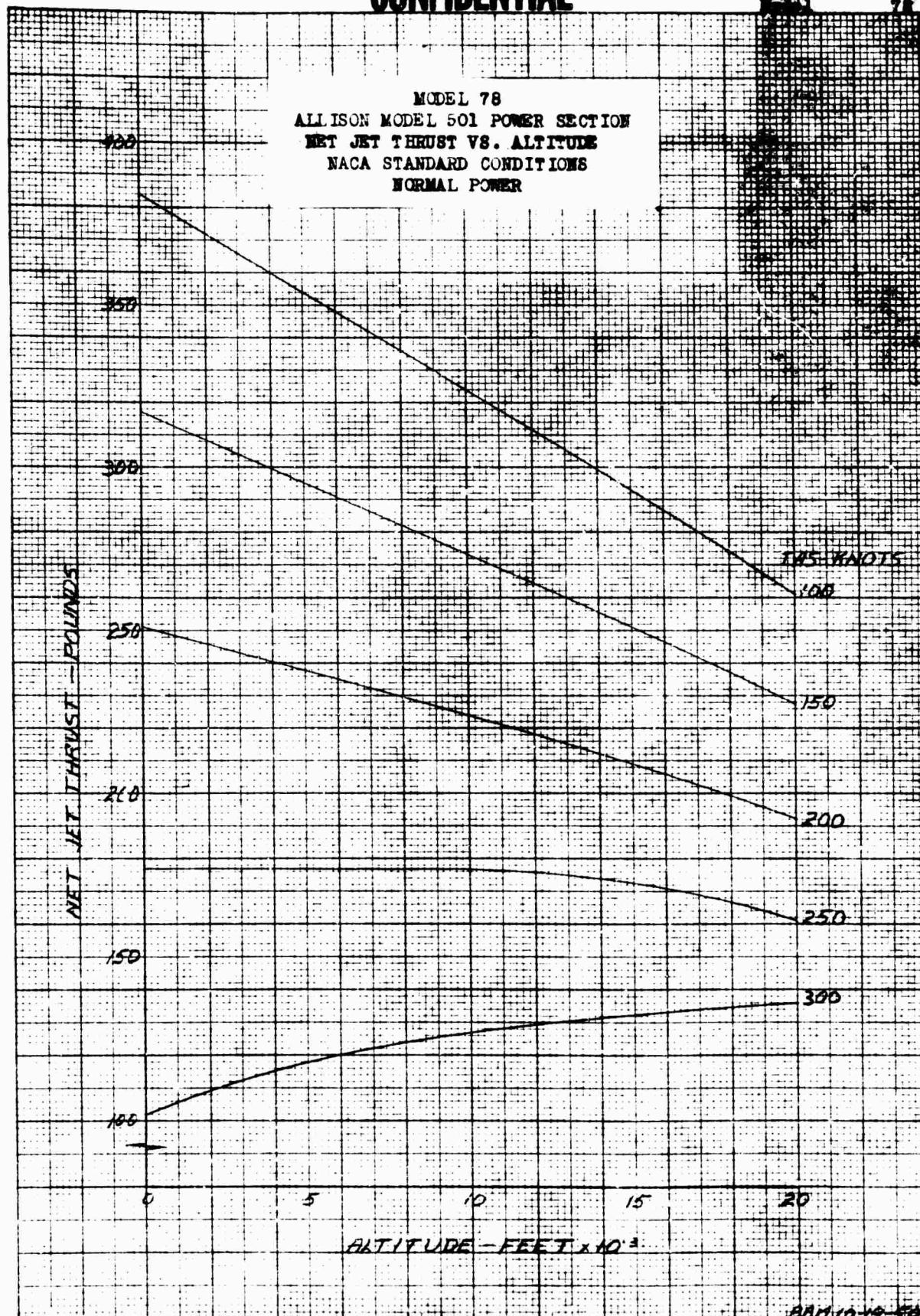
FIG 14



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KEUFFER, G. ESSER CO., N. Y. NO. 388-11  
18 X 18 in the 1/4 inch, 5/16 inch diameter  
MACHINING U.S.A.

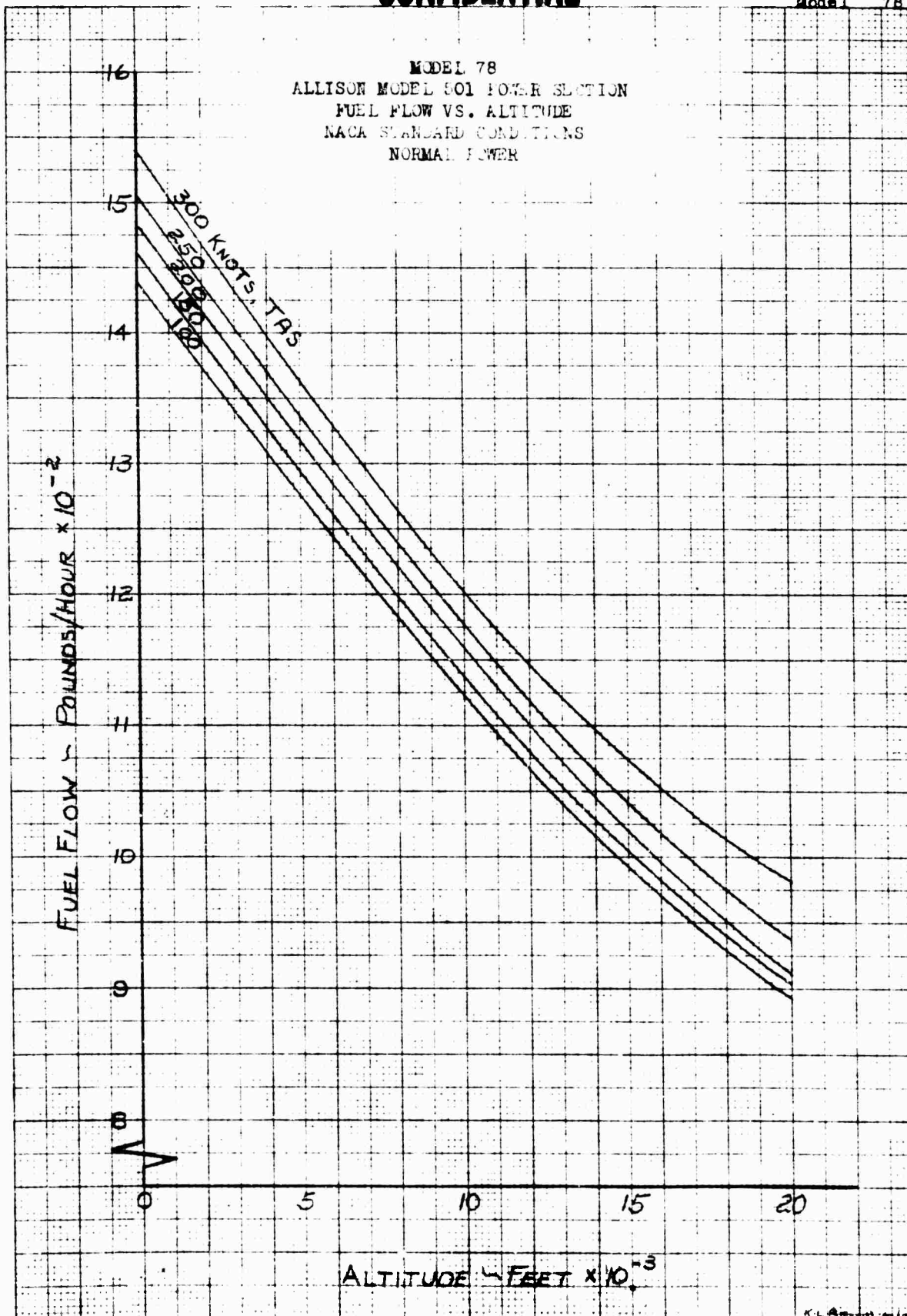
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FIG. 15

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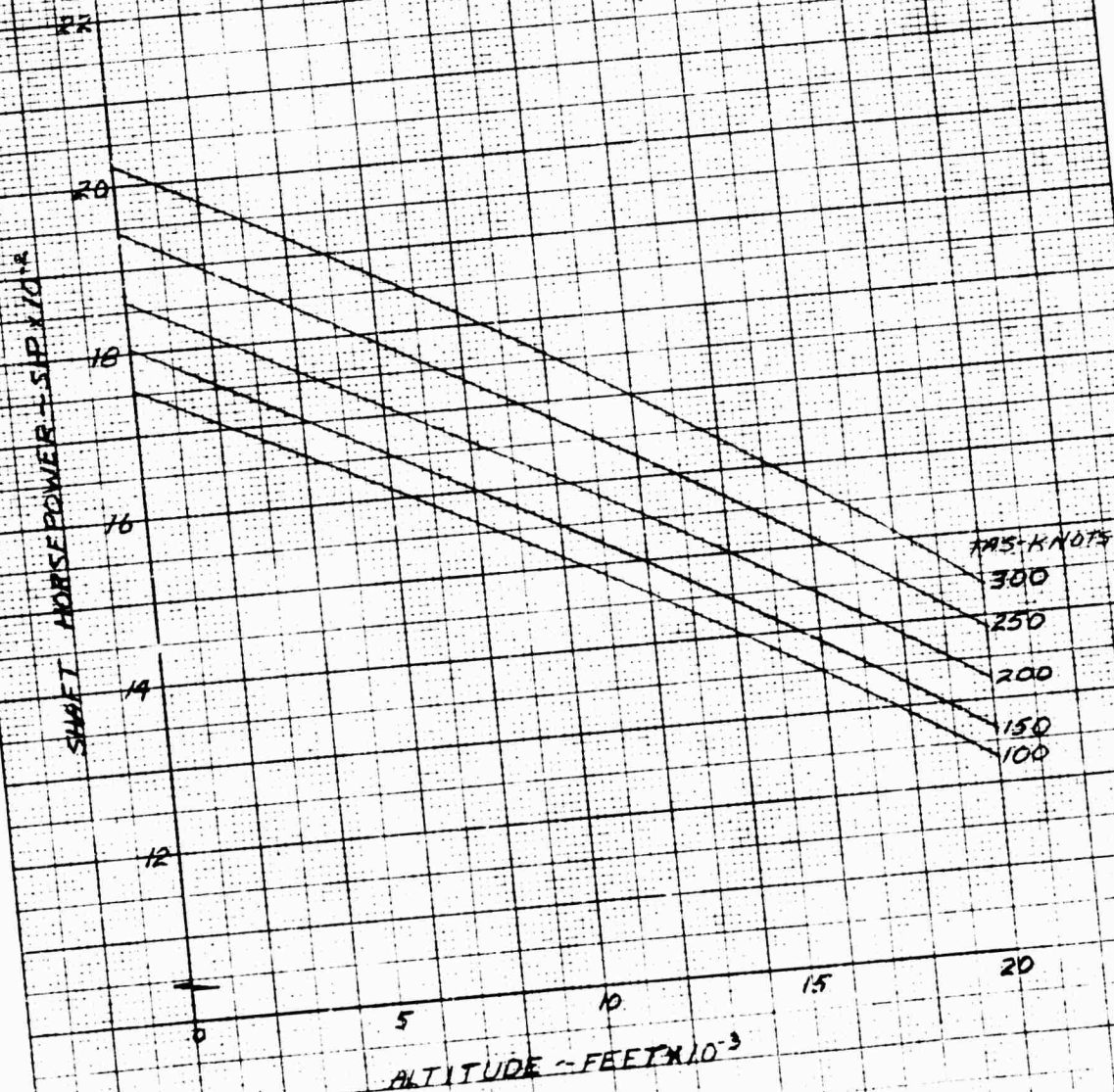
FIG. 16

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MODEL 78  
ALLISON MODEL 101 POWER SECTION  
SHAFT HORSEPOWER VS. ALTITUDE  
DAY, SUMMER DAY  
NORMAL POWER



KEUFFEL & ESSER CO., N. Y. NO. 128-11  
18 X 18 to the 1/2 inch. Six lines included  
MADE IN U.S.A.

REM 10-14-50

FIG 17.

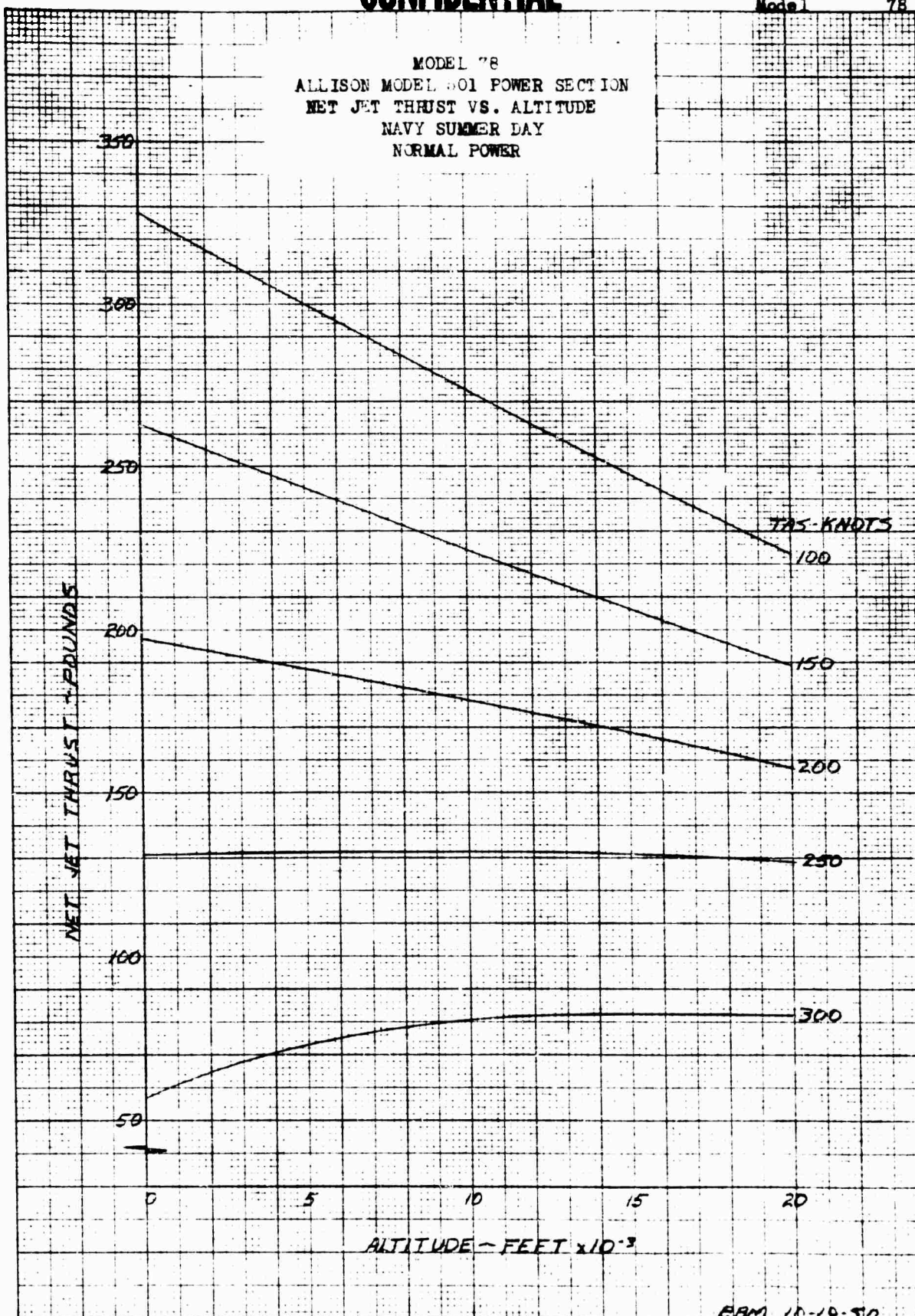
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EBM 10-10-510

FIG. 18

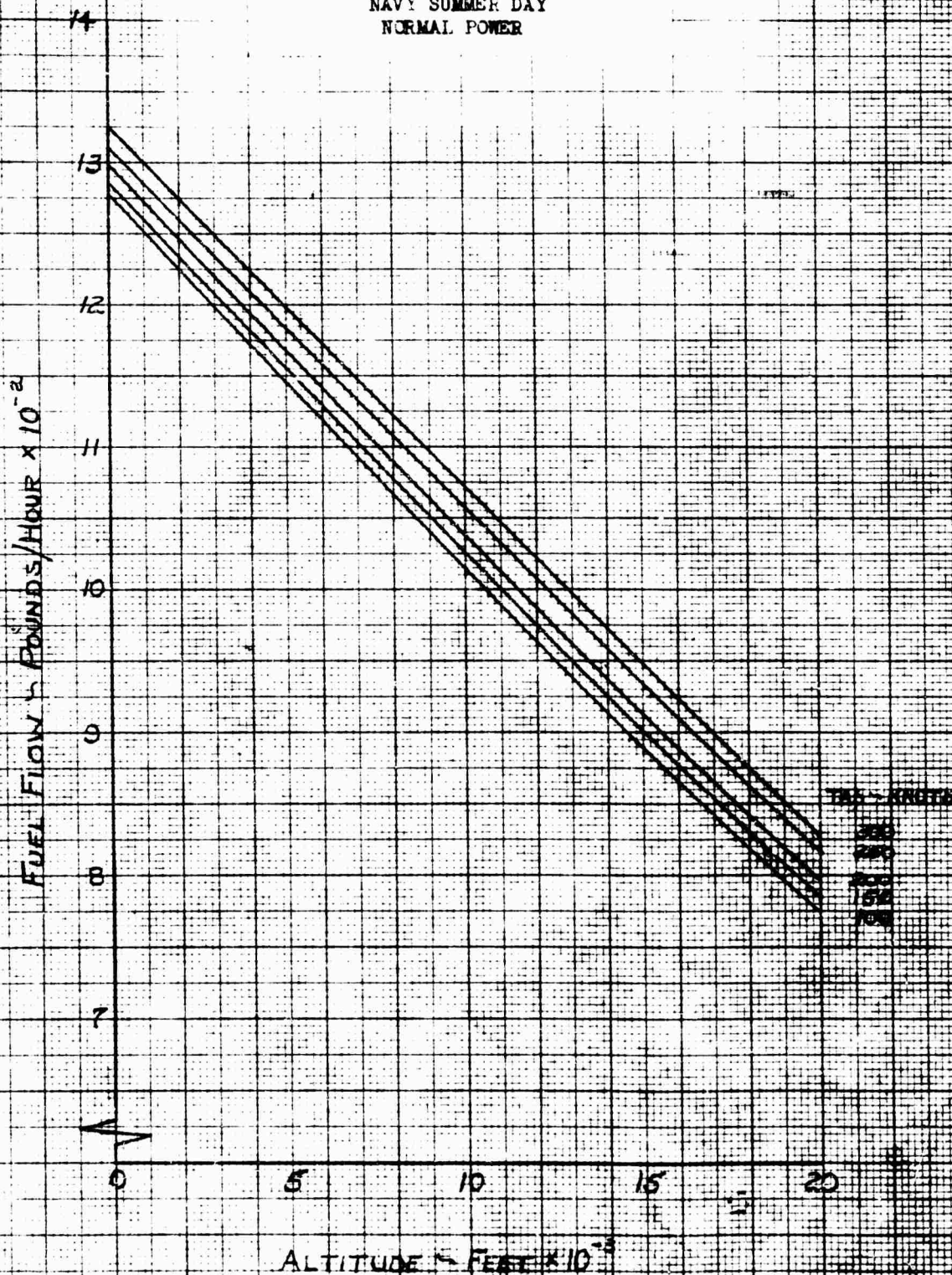
10 X 10 to the 1/2 inch 5th lines enclosed

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MODEL 78  
ALLISON MODEL 501 POWER SECTION  
FUEL FLOW VS. ALTITUDE  
NAVY SUMMER DAY  
NORMAL POWER



KEUFFEL & ESSER CO. N. Y. NO 28612  
10 x 10 to the 1/2 inch, 5 h. l. r.  
MADE IN U. S. A.

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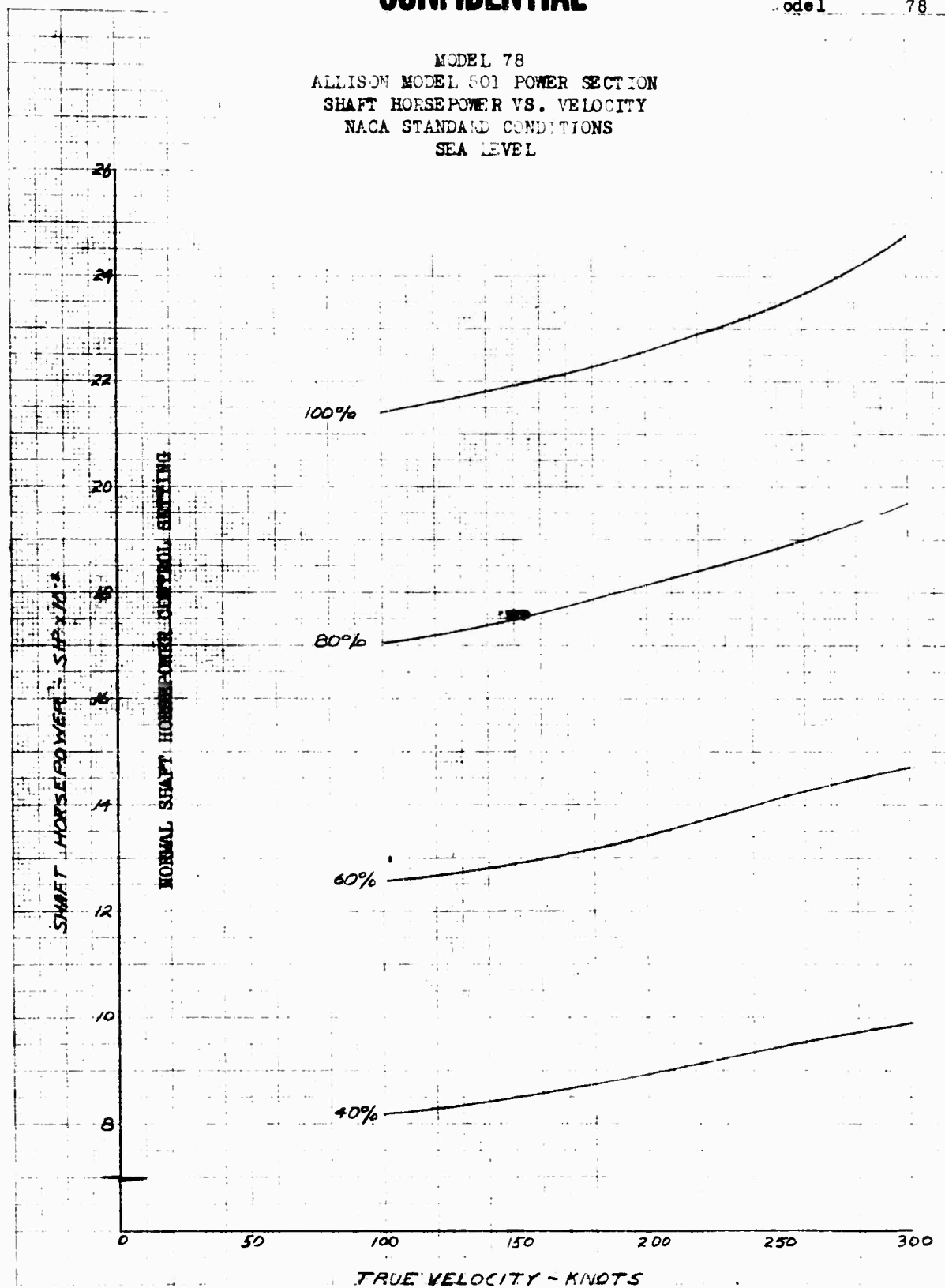
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MODEL 78  
ALLISON MODEL 501 POWER SECTION  
SHAFT HORSEPOWER VS. VELOCITY  
NACA STANDARD CONDITIONS  
SEA LEVEL



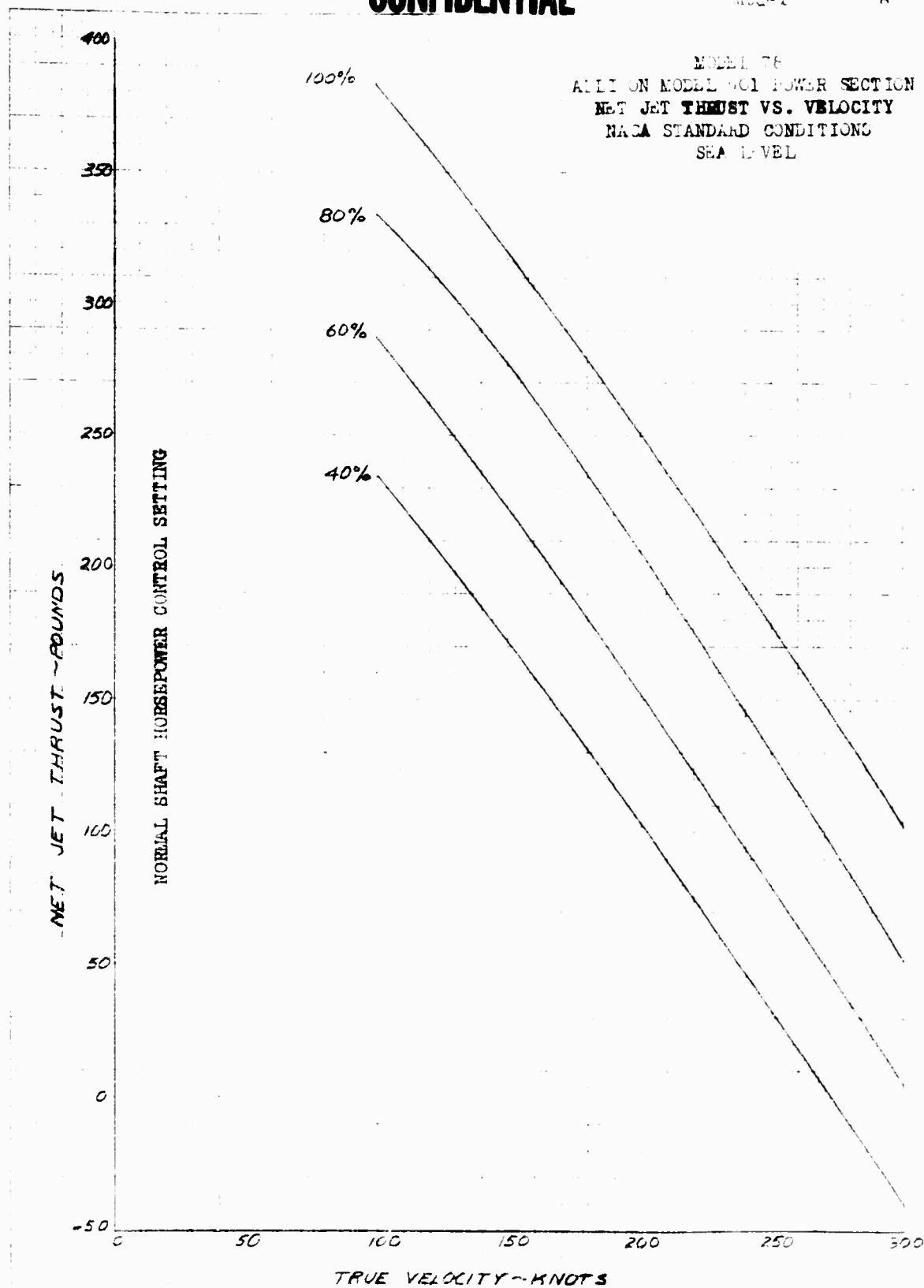
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BBM 10-19-50  
FIG 20

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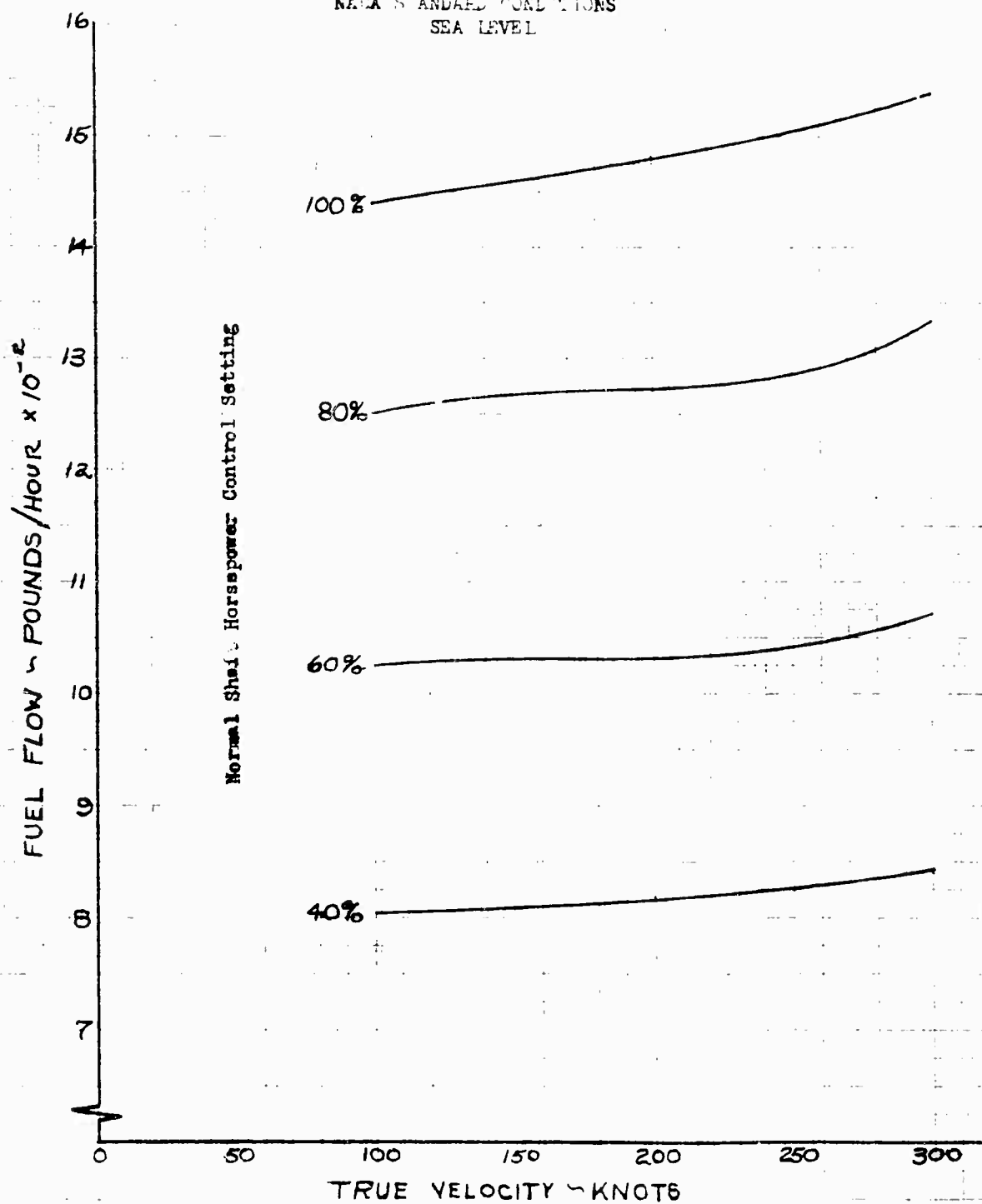
BBM 10-19-50  
FIG. 21

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MODEL 78  
ALLISON MODEL 501 POWER SECTION  
FUEL FLOW VS. VELOCITY  
NACA STANDARD CONDITIONS  
SEA LEVEL



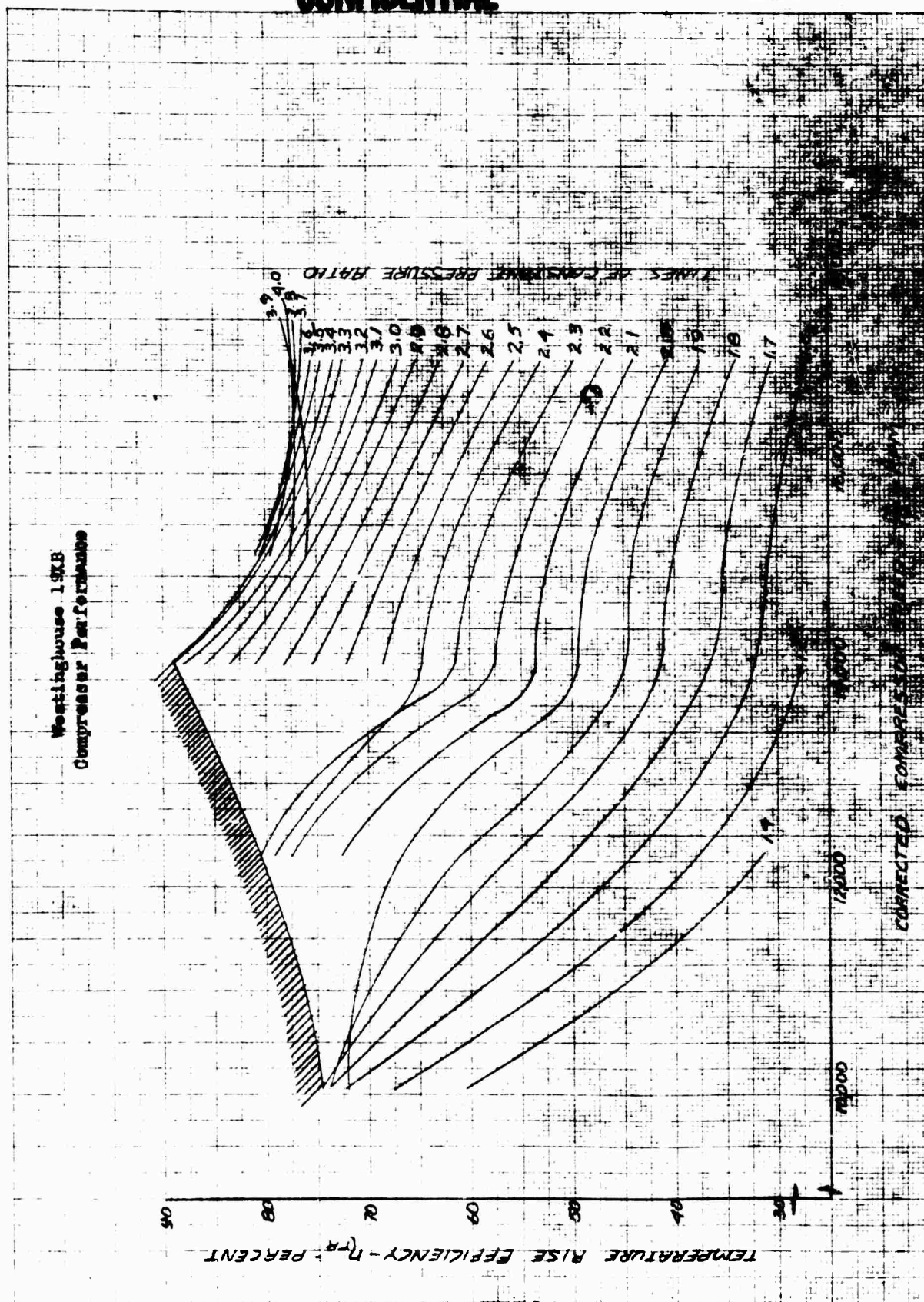
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KILBURN 10/11/50  
FIG. 22

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FIG. 23

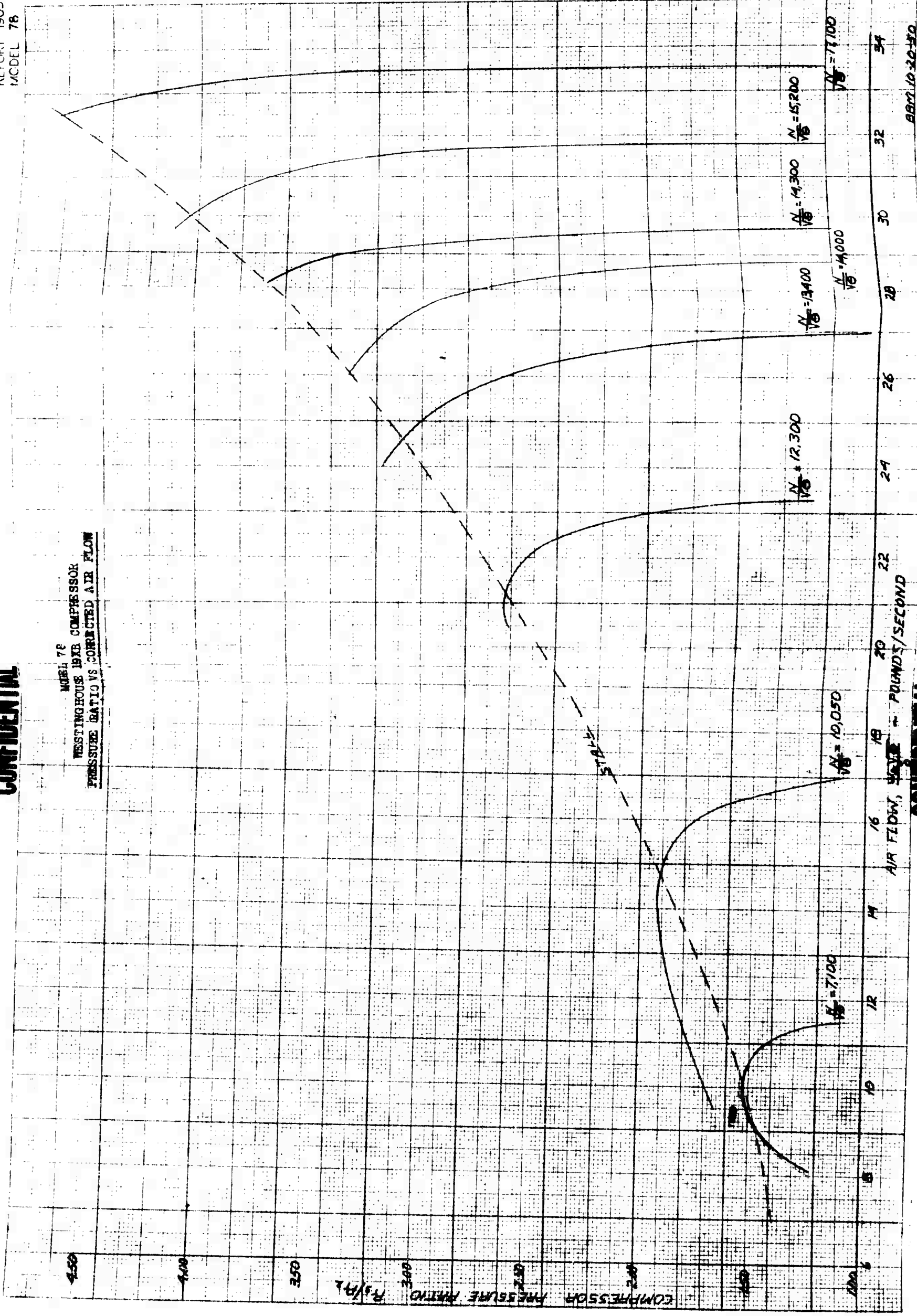


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MODEL 78  
WESTINGHOUSE 19XB COMPRESSOR  
PRESSURE RATIOS CORRECTED AIR FLOW



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FIG 24

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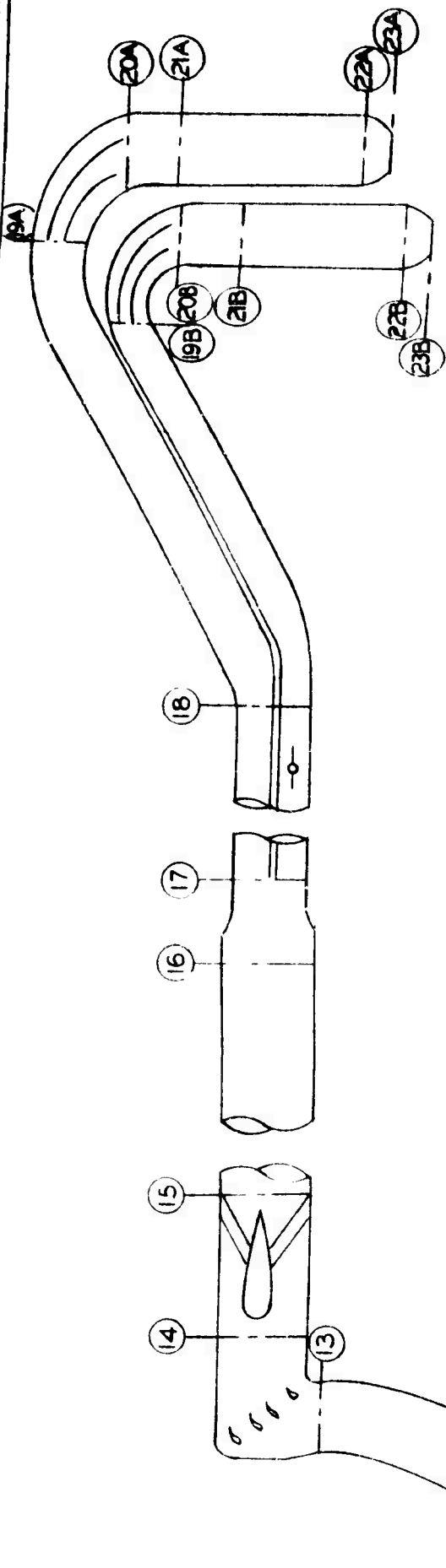
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MODEL 78  
PRESSURE JET DUCT SCHEMATIC

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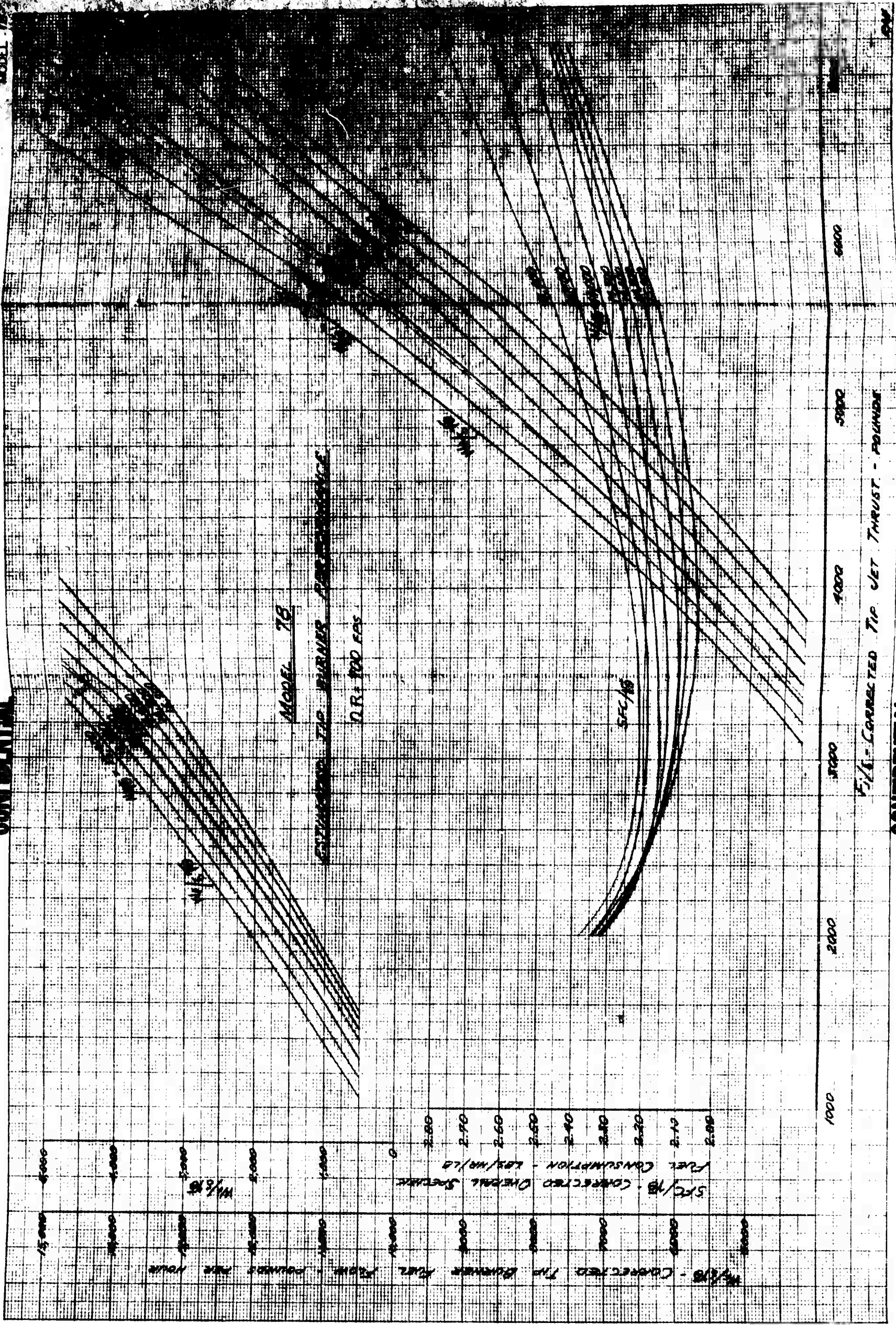
MAC 318 (REV 6-6-49)

FIG. 25





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FIG. 27

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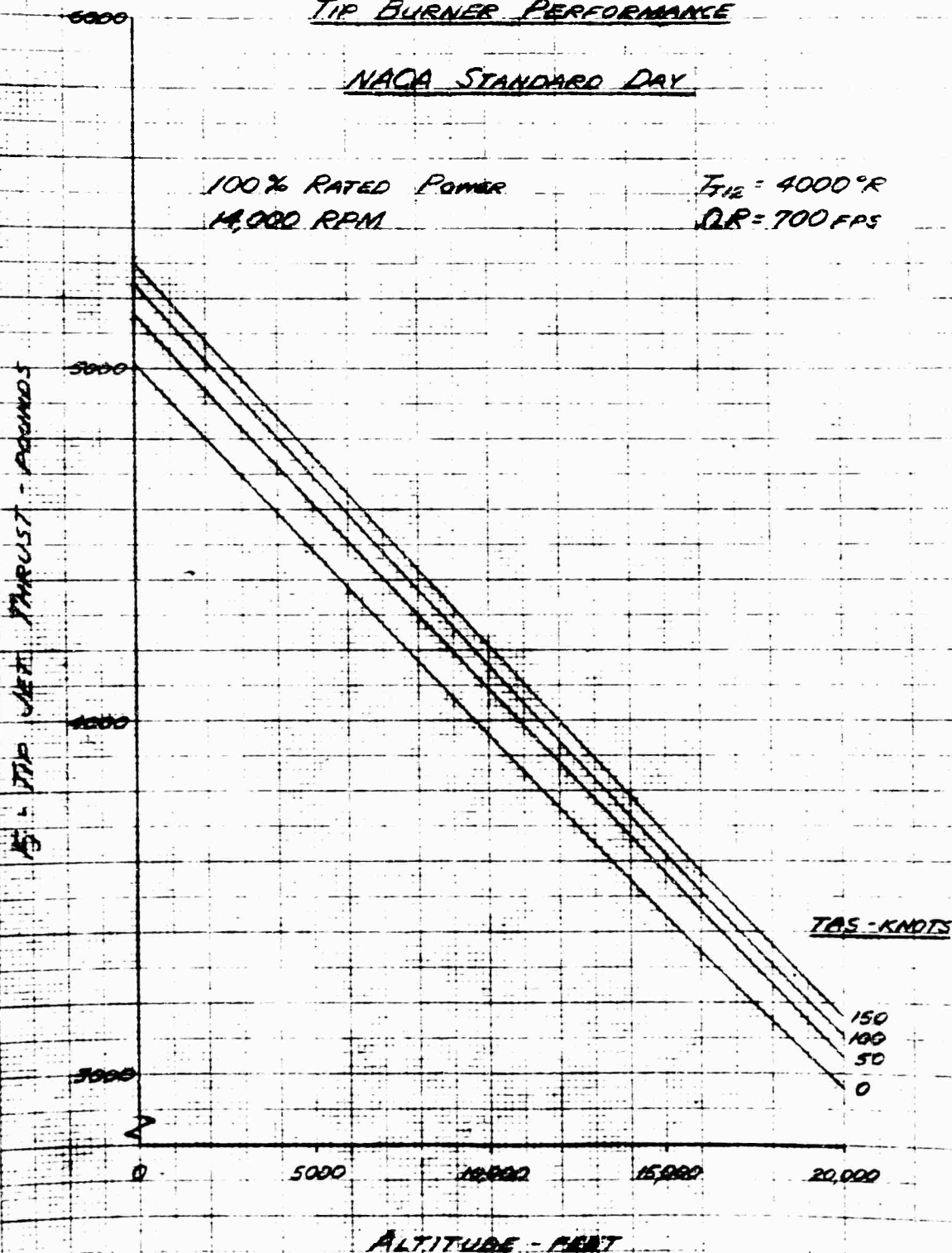
MODEL 78

TIP BURNER PERFORMANCE

NACA STANDARD DAY

100% RATED POWER  
14,000 RPM

$T_{12} = 4000^{\circ}\text{R}$   
 $Q_R = 700 \text{ FPS}$



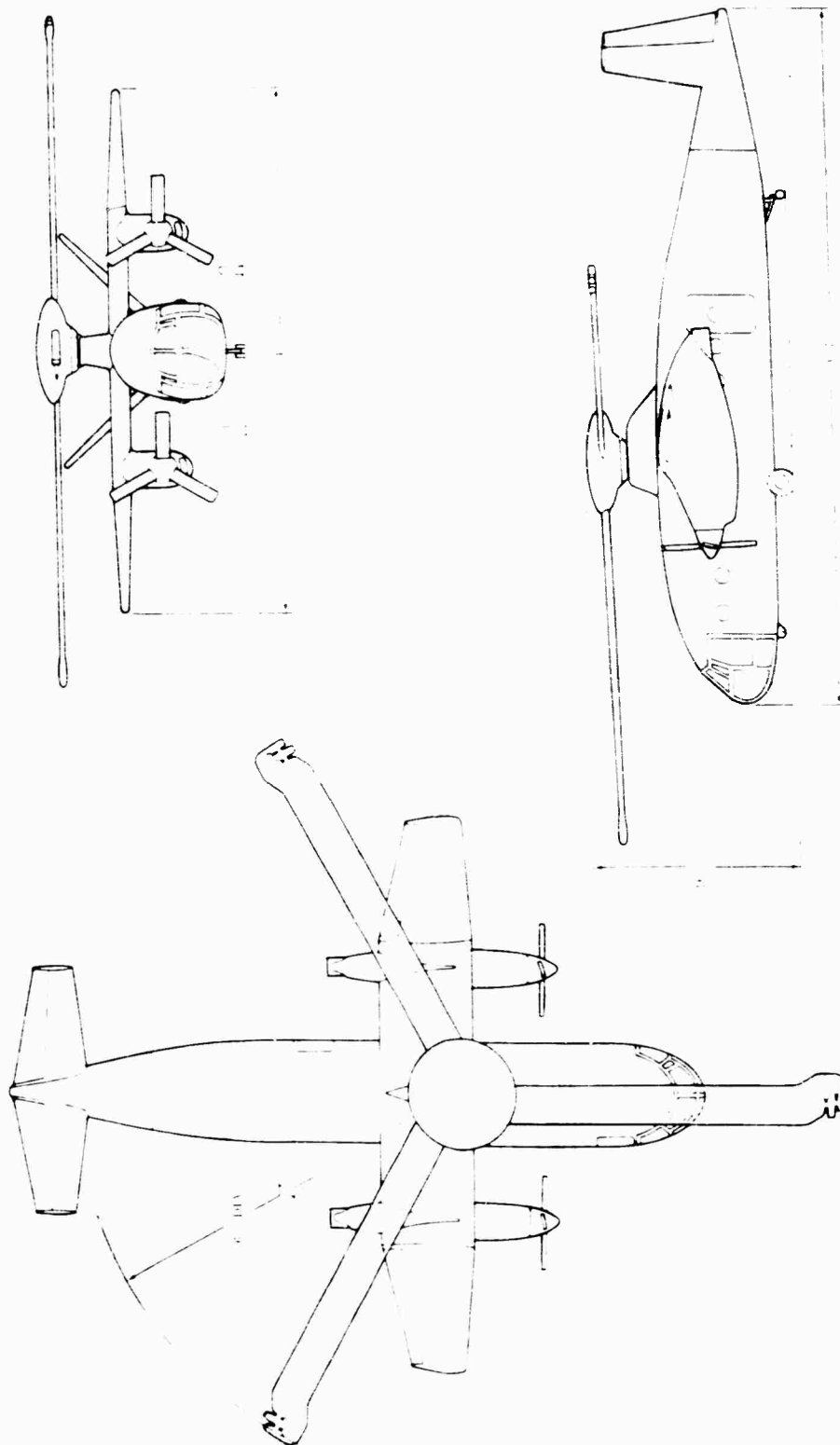
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RAH  
FIG. 28

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MODEL 78 GENERAL ARRANGEMENT

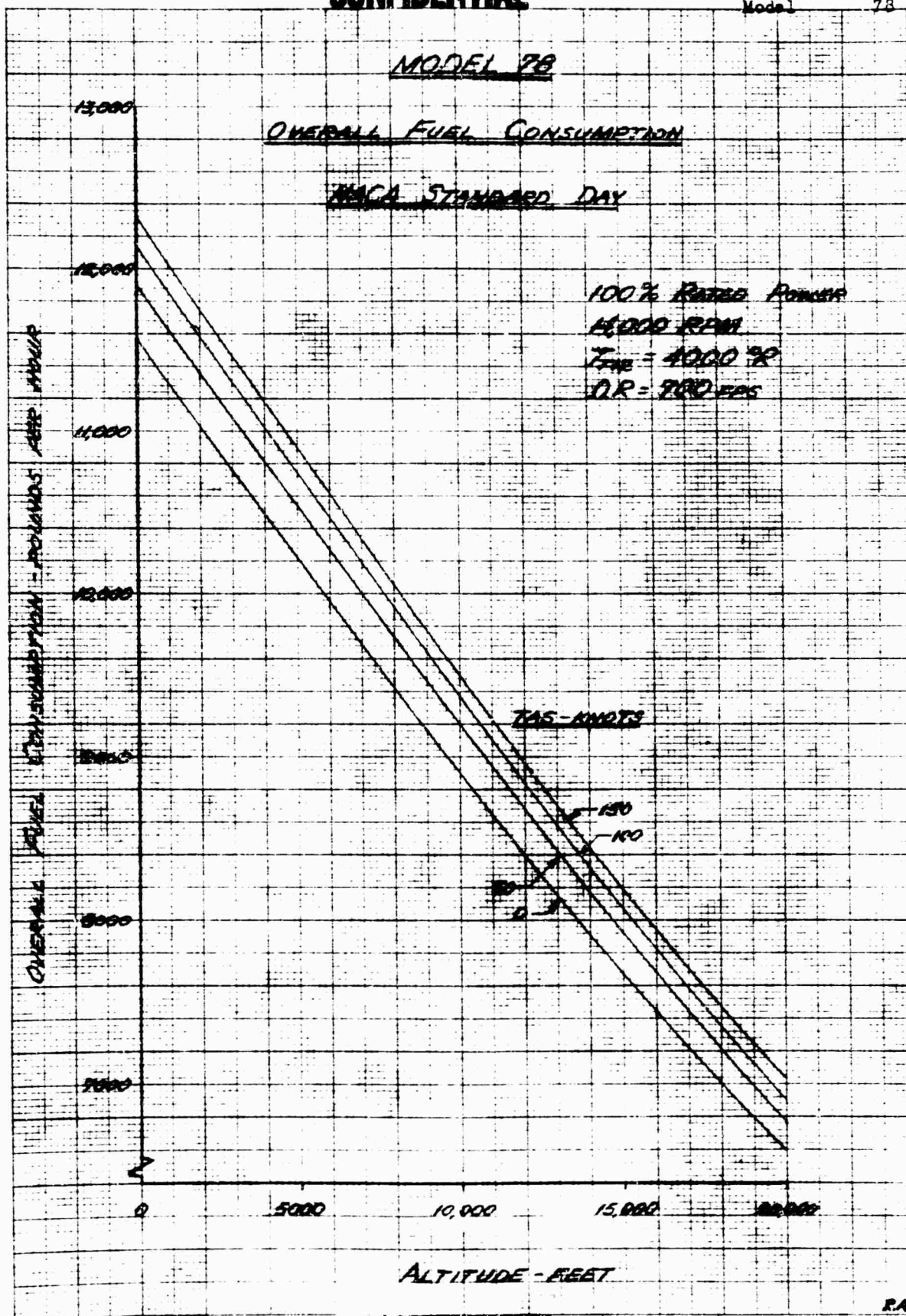
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KEUTCH & FOLK CO

NO. 359 11. 10 x 13 to the lines and 1/2 lines are used.  
Engraving, 10 x 10 in.

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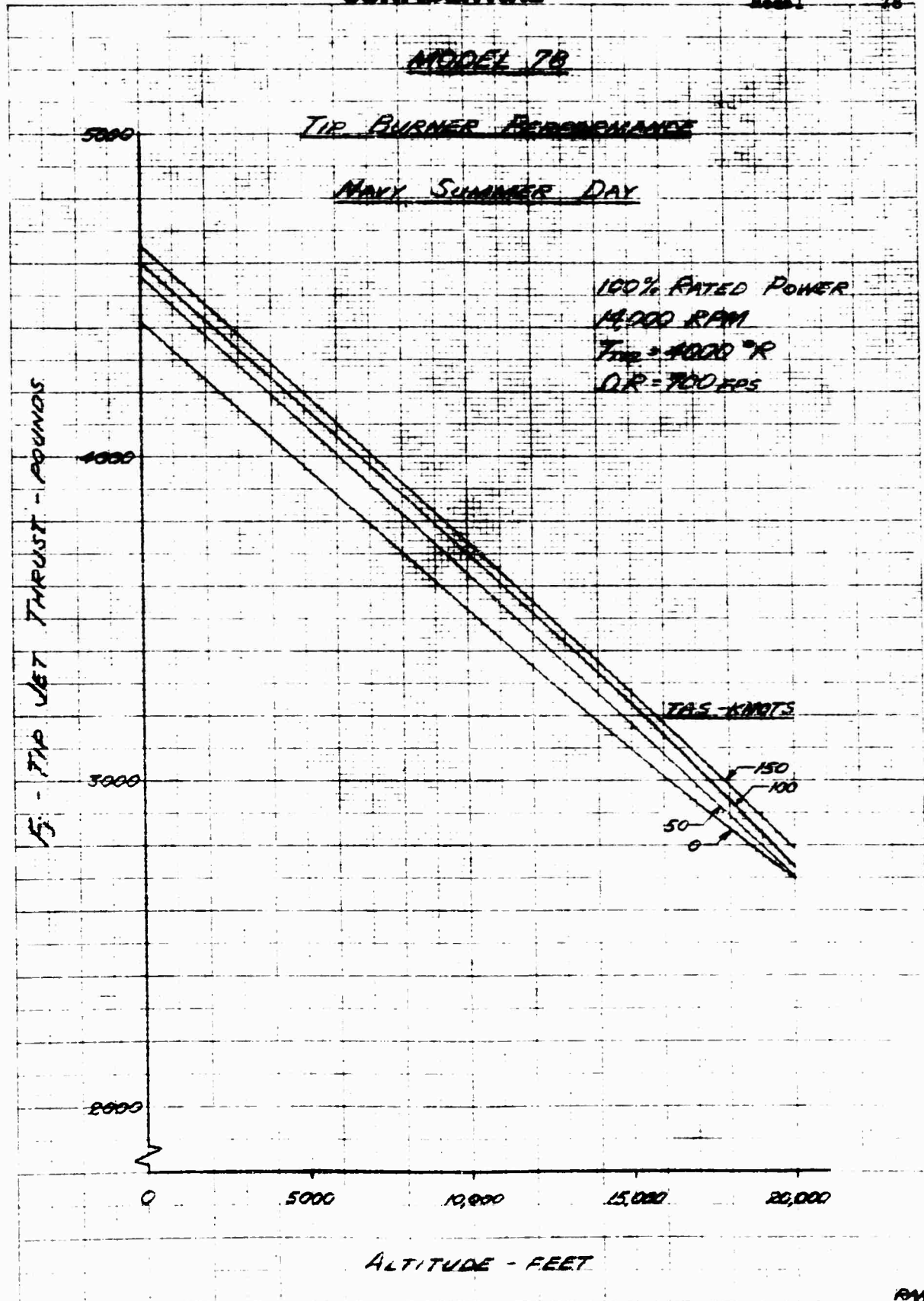
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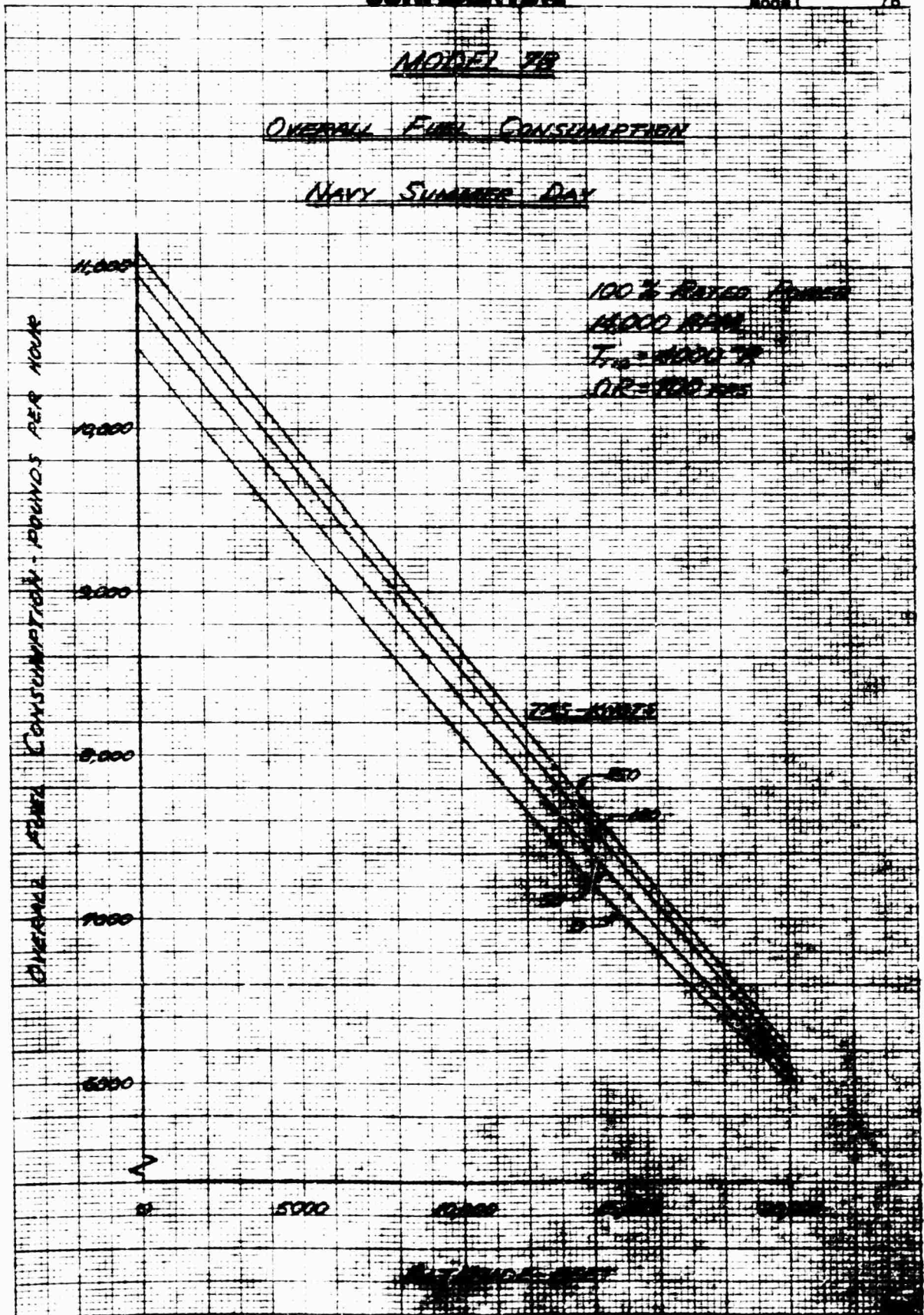
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FIG 30

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FIG. 31

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## ANALYSIS OF THE INLET DUCT ANALYSIS

1. NAME \_\_\_\_\_

100

STATION		1		2	
Total temperature, R		1.30	1.4	1.30	2
Static pressure, l./sq. in.		1.30	1.3	1.30	1.30
Density, slug/cu. ft.		1.30	1.30	1.30	1.30
Mass flow of air, lb./sec		1.30	1.30	1.30	1.30
Volume flow of air, cu. ft./sec		1.30	1.30	1.30	1.30
Cross sectional area sq. ft.		1.30	1.30	1.30	1.30
Velocity, ft./sec		1.30	1.30	1.30	1.30
$\frac{1}{2} \rho V^2$ l./sq. in.		1.30	1.30	1.30	1.30
Velocity of sound, ft./sec		1.30	1.30	1.30	1.30
Mach number		1.30	1.30	1.30	1.30
Compressibility factor		1.30	1.30	1.30	1.30
Impact pressure, l./sq. in.		1.30	1.30	1.30	1.30
Total pressure, l./sq. in.		1.30	1.30	1.30	1.30
Change in total pressure, l./sq. in.		1.30	1.30	1.30	1.30
Change in static pressure, l./sq. in.		1.30	1.30	1.30	1.30
Change in total temperature, R		1.30	1.30	1.30	1.30
Pressure-loss coefficient		1.30	1.30	1.30	1.30
Velocity parameter		1.30	1.30	1.30	1.30
Static temperature, °R		1.30	1.30	1.30	1.30
Air flow rate, lb./sec		1.30	1.30	1.30	1.30
Velocity, ft./sec		1.30	1.30	1.30	1.30
Temperature ratio		1.30	1.30	1.30	1.30
Pressure ratio $P_2/P_1$		1.30	1.30	1.30	1.30



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TABLE 2

ALLISON MODEL 501 ENGINE SECTION INLET DUCT ANALYSIS

Conditions:  $V_0 = 87$  Knots, Sea Level, NACA Standard Day, Normal Power

Quantity	STATION									
	0	1	1.1	1.30	1.45	1.60	2			
Total temperature, $^{\circ}\text{R}$	200.2	200.2	200.2	200.2	200.2	200.2	200.2			
Static pressure, $\text{lb./sq. in.}$	211	198.0	190.1	183.3	191.9	191.3	184.4			
Density, $\text{lb./cu. ft.}$	.002379	.0021	.00219	.00218	.00219	.00219	.00212			
Mass flow of air, $\text{lb./sec.}$	.014	.011	.019	.019	.019	.019	.019			
Volume flow of air, $\text{cu. ft./sec.}$	.009	.009	.019	.020	.020	.020	.020			
Cross-sectional area, $\text{sq. in.}$	1.11	1.11	1.11	1.11	1.29	1.29	1.11			
Velocity, $\text{ft./sec.}$	389	378	383	383	320	307	390			
$\frac{1}{2} \rho V^2$ , $\text{lb./sq. in.}$	20.0	17	17	18	116	110	161			
Velocity of sound, $\text{ft./sec.}$	1120	1110	1110	1108	1112	1112	1108			
Mach number	.131	.134	.134	.134	.292	.292	.302			
Compressor fuel, $\text{lb./hr.}$	1.004	1.004	1.003	1.001	1.022	1.022	1.032			
Impact pressure, $\text{lb./sq. in.}$	1.8	1.1	1.5	1.5	119	119	166			
Total pressure, $\text{lb./sq. in.}$	214.04	214.04	2052	2046	2036	2032	2010			
Change in total pressure, $\text{lb./sq. in.}$	$\Delta H$	0	-7	-16	-8	-6	-22			
Change in static pressure, $\text{lb./sq. in.}$	$\Delta P$	1.004	-82	-16	32	-6	-69			
Change in total temperature, $^{\circ}\text{R}$	$\Delta T_t$	0	0	0	0	0	0			
Pressure-loss coefficient	$\Delta H/q_c$			.1	.00	.00	.180			
Velocity, ft./hr.	$V_0$	87	87	87	87	87	87			
Static temperature, $^{\circ}\text{F}$	$T$	19.4	09	09	00	12	11			
Air flow rate, $\text{lb./hr.}$	$W_a$	29	29	29	29	29	29			
Velocity, $\text{ft./sec.}$	$V$	310	310	310	310	320	320			
Temperature ratio	$T_t/T$	1.02	1.02	1.02	1.02	1.016	1.017			
Pressure ratio	$P_t/P$	1.072	1.079	1.087	1.087	1.07	1.086			

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TABLE 3  
 AIR-ION MODEL 301 INLET SECTION INLET DUCT ANALYSIS

Conditions:  $V = 240$  knots,  $S = 100$  ft., NACA Standard La, Normal lower

Quantity	0	1	1.10	1.30	1.45	1.60	2
Total temperature, °R	211	2140	20.9	2040	2079	2072	1838
Static pressure, lb./sq. in., abs.	0.002378	0.00241	0.0022	0.00229	0.00233	0.00231	0.00211
Density, slugs/cu. ft.	0.0007	0.0007	0.0007	0.0007	0.0007	0.0007	0.0007
Mass flow of air, slugs/sec.	40	40	40	40	40	40	40
Volume flow of air, cu. ft./sec.	1.11	1.11	1.11	1.11	1.29	1.29	1.11
Cross sectional area, sq. ft.	360	360	360	360	326	329	418
Velocity, ft./sec.	194	182	170	170	124	125	185
$\frac{1}{2} \rho V^2$ lb./sq. ft.	1120	1120	1120	1120	1120	1120	1120
Velocity of sound, ft./sec.	0.320	0.320	0.344	0.289	0.293	0.293	0.374
Mach number	1.023	1.023	1.031	1.021	1.022	1.022	1.035
Compressibility factor	1.00	1.00	1.00	1.00	1.00	1.00	1.00
Impact pressure, lb./sq. ft.	2310	2310	2310	2310	2200	2200	2060
Total pressure, lb./sq. ft.	0	0	0	0	0	0	0
Change in total pressure, lb./sq. ft.	0	0	0	0	0	0	0
Change in static pressure, lb./sq. ft.	0	0	0	0	0	0	0
Change in total temperature °R	0	0	0	0	0	0	0
Pressure-less coefficient	0	0	0	0	0	0	0
Velocity parameter	0.654	0.654	0.654	0.654	0.654	0.654	0.654
Static temperature, °R	211	211	211	211	211	211	211
Air flow rate, lb./sec.	370	370	370	370	370	370	370
Velocity, ft./sec.	1.02	1.02	1.02	1.02	1.02	1.02	1.02
Temperature ratio	1.0094	1.0094	1.0094	1.0094	1.0094	1.0094	1.0094
Pressure ratio	1.0094	1.0094	1.0094	1.0094	1.0094	1.0094	1.0094

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TABLE 4  
SUMMARY OF ALLISON MODEL 501 POWER SECTION INLET DUCT ANALYSIS  
Sea Level, Normal Power

Condition		Static										$V_0 = 87 \text{ Knots}$		$V_0 = 240 \text{ Knots}$	
Velocity ratio $V_1/V_0$		$\infty$										2.11		0.903	
Angle of Attack $\alpha$		2.0										0		0	
Estimated $H/q_1$		1.11										0.816		1.54	
Estimate of losses through the duct															
Condition		Static										$V_0 = 87 \text{ Knots}$		$V_0 = 240 \text{ Knots}$	
Section	0-1	1-1.10	1.1-1.13	1.3-1.43	1.43-1.63	1.6-2	0-1	1-1.13	1.15-1.3	1.3-1.45	1.45-1.6	1.6-2	1.6-2		
$\Delta H/q_{local}$		.4800	.1040	.0516	.0522	.5620		.516	.1026	.0506	.0518	.190			
$q_{local}/q_0$	$\infty$	.9150	.9390	.9570	.7010	.7070	.159	.95	.968	.982	.721	.721	1.05		
$\Delta H/q_0$		.4390	.0973	.0494	.0356	.3900		.4920	.0994	.0497	.0373	.1370			
Total $H_{ot}/q_0$		1.0126												1.3791	
Over-all pressure loss and effect upon engine performance															
Condition		Static										$V_0 = 87 \text{ Knots}$		$V_0 = 240 \text{ Knots}$	
$\Delta H_{ot}/q_0$		1.0126										.8131		1.3791	
$\Delta H_{ot}/H_{ot}$		.0784										.0611		.11	
$\Delta SHP/SHF$		.126										.0362		.0602	
$\Delta F/F$		.122										.0791		.0769	
$\Delta W_F/W_F$		.0644										.0337		.0272	

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**CONFIDENTIAL**PAGE 55REPORT 1905MODEL 78**TABLE 5**WESTINGHOUSE 19XP COMPRESSOR - LIT. 1-3-1-10000

Condition: Static, Sea Level, ISA Standard Day, 1,000 ft.

Quantity	Symbol	0	1	1.1	2
1 Total temperature, °	$T_T$	518.4	518.4	518.4	518.4
2 Static pressure, lbs./sq.ft. abs.	$P$	2116	1782	1782	1782
3 Density, slug/cu.ft.	$\rho$	.001875	.001875	.001875	.001875
4 Mass flow of air, slug/sec.	$\dot{m}$	.714	.714	.714	.714
5 Volume flow of air, cu.ft./sec.	$Q$		330	330	330
6 Cross-sectional area, sq.ft.	$A$	.905	.905	.905	.905
7 Velocity, ft. sec.	$V$		330	330	330
8 $\rho V^2$ , lbs./sq.ft.	$q$		100	148	100
9 Velocity of sound, ft. sec.	$a$	1116	1116	1116	1116
10 Mach number	$M$		.316	.316	.316
11 Compressibility factor	$P_c$		1.027	1.0	1.031
12 Impact pressure, lbs./sq.ft.	$q P_c$		102.7	148	103.1
13 Total pressure, lbs./sq.ft.	$P_T$	2116	2116	2044	1980
14 Change in total pressure, lb./sq.ft.	$\Delta P_T$			-72	-136
15 Change in static pressure, lb./sq.ft.	$\Delta P$			-38	-138
16 Change in total temperature, °F	$\Delta T_T$				
17 Pressure-loss coefficient	$\Delta h/q$				
18 Velocity parameter	$R_{\rho a}/\rho q$	.647	.47	.47	.47
19 Static temperature, °F	$T$	518.4	500	500	518.4
20 Airflow rate, lb./sec.	$\dot{W}$	24.	21.5	24.5	24.
21 Velocity, ft. sec.	$V$		400	500	500
22 Temperature ratio	$T/T_0$		1.127	1.031	1.004
23 Pressure ratio	$P/P_0$		1.05	1.0	1.0

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TABLE 7 - ANALYSIS OF PRESSURE JET SYSTEM

Station		$N/\sqrt{e} = 14,000$						
		$P_{T3}/P_{T2} = 2.40$						
Qty.	Unit	0	3	4	5	6	7	8
$W_a$	#/sec		28.45	28.45	28.45	28.45	28.45	56.80
$T_T$	°R	518.4	740	740	740	740	740	740
$P_T$	PSIA	14.7	35.28	35.18	34.96	34.77	34.59	34.37
$A$	in <sup>2</sup>		187	100	100	100	100	200
$RW_a/P_{T1}$			.230	.431	.434	.436	.438	.441
$V$	ft/sec		170	328	331	333	334	337
$T$	°R		737.62	731.13	730.97	730.86	730.81	730.75
$\gamma$			1.3942	1.3944	1.3944	1.3944	1.3944	1.3945
$\gamma/\gamma - 1$			3.5368	3.5355	3.5355	3.5355	3.5355	3.5349
$T_T/T$			1.00323	1.01213	1.01235	1.01251	1.01258	1.01266
$P_T/P$			1.01147	1.04360	1.04440	1.04487	1.04512	1.04540
$P$	PSIA		34.88	33.71	33.47	33.28	33.10	32.88
$q_0$	PSI		0.40	1.47	1.49	1.49	1.49	1.49
$f$	ft		2.95*	2.95	2.95	2.95	2.95	4.767
$W_a/g_P$			.300	.300	.300	.300	.300	.371
$RN$			$2.45 \times 10^6$	$2.45 \times 10^6$	$2.45 \times 10^6$	$2.45 \times 10^6$	$2.45 \times 10^6$	$3.1 \times 10^6$
$f$					.0031			
$1/D$					10.15			
$\Delta H/q$			.25	.15	.128	.12	.10	.05
$\Delta H$			.10	.22	.19	.13	.22	.07
$V_{Tx}$								
$V_{Ty}$								
$V_{Ty} - V_{Tx}$								
$P_{Ty}/P_{Tx}$								
$\Delta H$								

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TABLE 7 Continued

Qty.	Station Unit	9	10	11	12	13	14	15
$W_a$	#/sec	56.90	56.90	56.90	56.90	18.97	18.97	18.97
$T_T$	$^{\circ}R$	740	740	740	740	740	740	740
$P_T$	PSIA	34.30	34.15	33.87	33.83	33.80	33.53	33.23
$A$	$in^2$	200	209	410	251	78.4	44.2	44.2
$RW_a/P_{TA}$		.442	.425	.218	.357	.382	.682	.688
$V$	ft/sec	338	323	162	268	288	551	557
$T$	$^{\circ}R$	730.58	731.40	737.84	734.08	733.17	715.96	714.41
$\gamma$		1.3945	1.3944	1.3941	1.3943	1.3944	1.3950	1.3950
$\gamma/\gamma - 1$		3.5349	3.5355	3.5374	3.5361	3.5355	3.5316	3.5316
$T_T/T$		1.01289	1.01176	1.00293	1.00806	1.00932	1.03358	1.03582
$P_T/P$		1.04632	1.04215	1.01040	1.02876	1.03334	1.1238	1.1323
$P$	PSIA	32.78	32.77	33.52	32.88	32.71	29.84	29.35
$q_c$	PSI	1.52	1.38	0.35	0.95	1.09	3.69	3.88
$\rho$	ft	4.767			14.45			1.96
$W_a/g_p$		.371			.122			.300
$RN$		$3.1 \times 10^6$			$1.1 \times 10^6$			$2.45 \times 10^6$
$f$					.0035			.0031
$1/D$					1.90			11.3
$\Delta H/q$		.10	.20	.10	.027	.25	.10	.14
$\Delta H$		.15	.28	.04	.03	.27	.37	.54
$V_{T_x}$							0	75
$V_{T_y}$							75	220
$V_{T_y} - V_{T_x}$							5625	42,775
$P_{T_y}/P_{T_x}$							1.00223	1.01698
$\Delta H$								.07

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TABLE 7 Continued

Qty.	Station Units	16	17a	18a	19a	20a	21a	22a
W <sub>a</sub>	#/sec	18.97	9.48	9.48	9.49	9.48	9.48	
T <sub>T</sub>	°R	740	740	740	740	740		
P <sub>T</sub>	PSIA	33.25	33.25	33.01	33.85	33.84	33.54	33.22
A	in <sup>2</sup>	44.2	24.3	24.3	24.3	48.4		
RW <sub>a</sub> /P <sub>T</sub> A		.688	.625	.577	.564	.284		
V	ft/sec	557	498	453	441	212		
T	°R	714.41	719.55	723.08	723.97	736.30		
γ		1.3950	1.3948	1.3947	1.3947	1.3943		
γ/γ - 1		3.5316	3.5329	3.5336	3.5336	3.5361		
T <sub>T</sub> /T		1.03582	1.02842	1.02340	1.02214	1.00503		
P <sub>T</sub> /P		1.1323	1.1040	1.0850	1.08035	1.01790		
P	PSIA	29.37	30.12	33.19	34.11	36.19		
q <sub>c</sub>	PSI	3.88	3.13	2.82	2.74	.65		
P	ft		17.08					
W <sub>a</sub> /q <sub>p</sub>			.172					
RN			1.45x10 <sup>6</sup>					
f			.0035					
1/D			44.0					
ΔH/q		.001	.58	.15	.15	2.0		
ΔH		0	1.82	.42	.41	1.30	.32	
V <sub>Tx</sub>			220	618	680			
V <sub>Ty</sub>			618	680	700			
V <sub>Ty</sub> - V <sub>Tx</sub>			333,524	80,476	27,600			
P <sub>Ty</sub> /P <sub>Tx</sub>			1.1378	1.0350	1.01089			
ΔH	.56 0		4.58	1.26	.40			

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TABLE 7 Continued

Station	23a	17b	18b	19b
Qty. Units				
$\dot{W}_a$ #/sec		9.48	9.48	9.48
$T_e$ °R	1885	740	740	740
$P_T$ PSIA	35.22	33.25	35.89	36.62
$A$ in <sup>2</sup>	22.51	22.1	22.1	22.1
$R\dot{W}_a/P_T A$		.688	.637	.624
$V$ ft/sec	1932	557	507	.497
$T$ °R	1594	714.41	718.80	719.63
$\gamma$		1.3950	1.3949	1.3948
$\gamma/\gamma - 1$		3.5316	3.5323	3.5329
$T_T/T$		1.03582	1.02949	1.02831
$P_T/P$		1.1323	1.1060	1.1034
$P$ PSIA		29.37	32.39	33.19
$q_c$ PSI		3.88	3.50	3.43
$\rho$ ft		1.39		
$\dot{W}_a/\dot{E}P$		2.11		
$RM$		$1.65 \times 10^6$		
$f$		.0032		
$1/D$		39.34		
$\Delta H/q$		.50	.15	.15
$\Delta H$		1.94	.53	.51
$V_{T_x}$				
$V_{T_y}$				
$V_{T_y} - V_{T_x}$				
$P_{T_e}/P_{T_x}$		1.1378	1.0350	1.01089
$\Delta H$		4.08	1.25	.40

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TABLE 7 Continued

Station		20b	21b	22b	23b
Qty.	Unit				
$W_a$	$\#/\text{sec}$	9.48	9.48		
$T_T$	$^{\circ}\text{R}$	740			1865
$P_T$	PSIA	36.81	35.21	34.91	34.91
$A$	$\text{in}^2$	48.4			22.51
$RW_a/F_{TA}$		.283			
$V$	$\text{ft}/\text{sec}$	213			1922
$T$	$^{\circ}\text{R}$	736.26			1576
$\gamma$		1.3943			
$\gamma/\gamma-1$		3.8361			
$T_T/T$		1.00508			
$P_T/P$		1.01808			
$P$	PSIA	35.86			
$q_0$	PSI	.65			
$P$	$\text{ft}$				
$W_a/q_F$					
$RN$					
$1/D$					
$\Delta H/q$			2.0		
$\Delta H$			1.30	.30	
$V_{Tx}$					
$V_{Ty}$					
$V_{Ty}-V_{Tx}$					
$P_{Ty}/P_{Tx}$					
$\Delta H$					

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TABLE 7 Continued

Duct		a	b	Total Model 78
Qty.	Unit			
A	ft <sup>2</sup>	.1563	.1503	469
P <sub>T22</sub>	PSIA	35.22	34.91	
$\gamma$		1.3649	1.3664	
$\gamma/\gamma - 1$		3.7405	3.7293	
$(2/\gamma + 1)^{\gamma-1}$		.5345	.5345	
P <sub>23</sub>	PSIA	18.83	18.66	
W <sub>E</sub>	#/sec	9.67	9.61	
T <sub>23</sub>	°R	1594	1586	
T <sub>T</sub>	°R	1885	1865	1875
C <sub>pav</sub>		.2575	.2571	
$\Delta T$	°R	858	840	
W <sub>F</sub>	#/sec	.12931	.12637	.276704
W <sub>F</sub>	#/hr	468.5	454.9	2761
F/A		.01364	.01333	
V	ft/sec	1932	1922	
$\eta \frac{W_q}{g} \Delta V$	#	350	348	
( $\Delta P$ )A	#	93	89	
F <sub>J</sub>	#	443		2634
H <sub>req</sub>				4290
HP <sub>out put/engine</sub>				2283
W <sub>FE</sub> /engine				1480
W <sub>FE</sub> total				2960
W <sub>FE</sub> + W <sub>FJ</sub>				3721
Overall SFC				2.17

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